

ORTHOGONALITY PRINCIPLE:

The orthogonality principle states that the principle modes of vibration for a mechanical system with more than one degree of freedom occurs along mutually perpendicular straight lines.

The orthogonality principle can be very useful for calculating the system's natural frequency. For a two degree of freedom system orthogonality principle may be written as.

$$m_1 A_1 A_2 + m_2 B_1 B_2 = 0$$

A_1, A_2, B_1 and B_2 are the vibration amplitude of the coordinates for the first and second vibration modes.

For a three degree of freedom, orthogonality principle may be written as.

$$m_1 A_1 A_2 + m_2 B_1 B_2 + m_3 C_1 C_2 = 0$$

$$m_1 A_2 A_3 + m_2 B_2 B_3 + m_3 C_2 C_3 = 0$$

$$m_1 A_1 A_3 + m_2 B_1 B_3 + m_3 C_1 C_3 = 0.$$

EIGEN VALUES AND EIGEN VECTORS:-

From the matrix form of equation of motion,

$$m_1 \ddot{x}_1 + (k_1 + k_2)x_1 - k_2 x_2 = 0$$

$$m_2 \ddot{x}_2 - k_1 x_1 + (k_1 + k_2)x_2 - k_2 x_3 = 0$$

$$m_3 \ddot{x}_3 - k_2 x_2 + k_2 x_3 = 0.$$

$$[M] \{\ddot{x}\} + [K] \{x\} = 0$$

$$[M][M]^{-1} = [I]$$

Unit matrix

$$[I] \{\ddot{x}\} + [C] \{x\} = 0$$

$$[K][M]^{-1} = [C]$$

dynamic matrix.

$$x = A \sin \omega t$$

$$\ddot{x} = -\omega^2 A \sin \omega t$$

$$= -\lambda A \sin \omega t$$

$\lambda = \omega^2 =$ eigen value.

$A =$ eigen vectors,

$$\begin{Bmatrix} A_1 \\ A_2 \\ A_3 \end{Bmatrix}$$

$$\begin{Bmatrix} \lambda_1 \\ \lambda_2 \\ \lambda_3 \end{Bmatrix}$$

$$\{ \lambda [I] - [C] \} \{ A \} = 0$$

$$\lambda_1 = 0.198 \frac{k}{m}$$

$$\frac{0.198k}{m} \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \frac{k}{m} \begin{bmatrix} 2 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 1 \end{bmatrix} \begin{Bmatrix} A_1 \\ A_2 \\ A_3 \end{Bmatrix} = 0$$

$$\frac{k}{m} \begin{bmatrix} -1.802 & 1 & 0 \\ 1 & -1.802 & 1 \\ 0 & 0.802 & 0 \end{bmatrix} \begin{Bmatrix} A_1 \\ A_2 \\ A_3 \end{Bmatrix} = 0$$

$$-1.802 A_1 + A_2 = 0$$

$$A_1 - 1.802 A_2 + A_3 = 0$$

$$A_2 - 0.802 A_3 = 0$$

mode shape will be $\begin{Bmatrix} A_1 \\ A_2 \\ A_3 \end{Bmatrix}$

$$\frac{A_2}{A_1} = 1.802$$

$$\frac{A_3}{A_1} = 2.247$$

First mode shape will be

$$A_{11} = A_1 \begin{Bmatrix} 1.0 \\ 1.802 \\ 2.247 \end{Bmatrix}$$

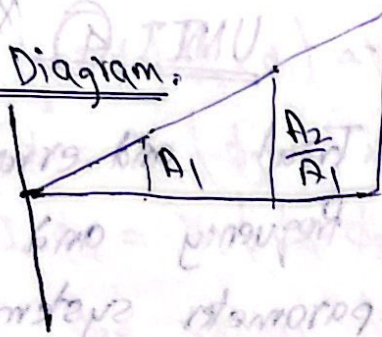
$$\lambda_2 = 1.555 \frac{k}{m}$$

$$A_{21} = A_2 \begin{Bmatrix} 1.0 \\ 0.445 \\ -0.8019 \end{Bmatrix}$$

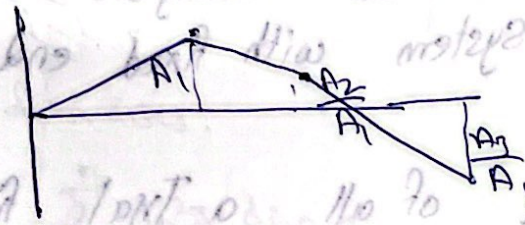
$$\lambda_3 = 5.247 \frac{k}{m}$$

$$A_{31} = A_3 \begin{Bmatrix} 1.0 \\ 1.247 \\ 0.555 \end{Bmatrix}$$

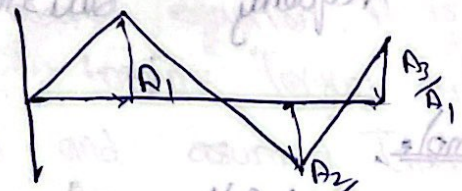
Mode Shape Diagram



First mode



Second mode



Third mode

Result :-

$$\lambda_1 = 0.198 \frac{K}{M}$$

$$\lambda_2 = 1.555 \frac{K}{M}$$

$$\lambda_3 = 3.247 \frac{K}{M}$$

$$A_{1i} = A_1 \begin{Bmatrix} 1.0 \\ 1.802 \\ 2.247 \end{Bmatrix}$$

$$A_{2i} = A_2 \begin{Bmatrix} 1.0 \\ 0.445 \\ -0.8019 \end{Bmatrix}$$

$$A_{3i} = A_3 \begin{Bmatrix} 1.0 \\ 1.247 \\ 0.555 \end{Bmatrix}$$

