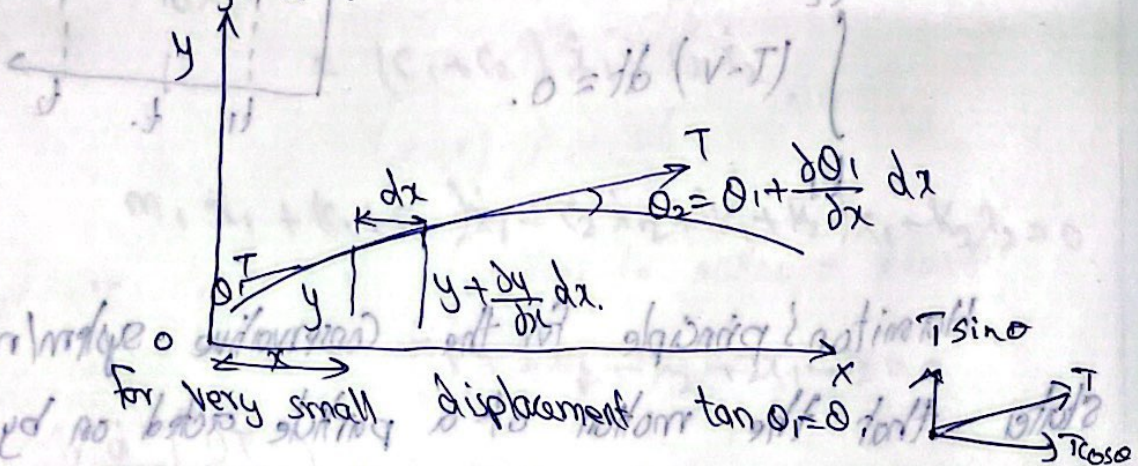


LATERAL VIBRATIONS OF A STRING.

Consider a vibrating string of mass e per unit length having transverse vibrations under tension T as shown.

It is assumed that for a very small amplitude of string vibration the tension T remains constant throughout.



$$\tan \theta_1 = \frac{dy}{dx}$$

$$\theta_1 = \frac{dy}{dx}$$

$$\theta_2 = \frac{dy}{dx} + \frac{d}{dx} \left(\frac{dy}{dx} \right) dx$$

$$\theta_2 = \theta_1 + \frac{d\theta_1}{dx} dx$$

Resolving Tension along y-axis.

$$T \sin \left[\theta_1 + \frac{d\theta_1}{dx} dx \right] - T \sin \theta_1 = \text{mass} \times \text{acceleration}$$

$$= e dx \frac{d^2 y}{dt^2}$$

$$T \left(\theta_1 + \frac{d\theta_1}{dx} dx \right) - T \theta_1 = e dx \frac{d^2 y}{dt^2}$$

$$T \frac{d\theta_1}{dx} dx = e dx \frac{d^2 y}{dt^2}$$

$$\frac{T}{\rho} \frac{\partial \theta_1}{\partial x} = \rho \frac{\partial^2 y}{\partial t^2}$$

$$\frac{T}{\rho} \left(\frac{\partial \theta_1}{\partial x} \right) = \frac{\partial^2 y}{\partial t^2}$$

$$\theta_1 = \frac{\partial y}{\partial x}$$

$$\frac{T}{\rho} \frac{\partial}{\partial x} \left(\frac{\partial y}{\partial x} \right) = \frac{\partial^2 y}{\partial t^2}$$

$$\frac{T}{\rho} \left(\frac{\partial^2 y}{\partial x^2} \right) = \frac{\partial^2 y}{\partial t^2}$$

Assuming $a^2 = \frac{T}{\rho}$, the above equation can be written as.

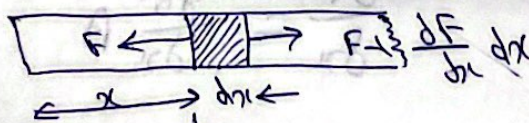
$$a^2 \left(\frac{\partial^2 y}{\partial x^2} \right) = \frac{\partial^2 y}{\partial t^2}$$

$$\frac{\partial^2 y}{\partial x^2} = \frac{1}{a^2} \frac{\partial^2 y}{\partial t^2}$$

This is one-dimensional wave equation for lateral vibrations of string.

LONGITUDINAL VIBRATIONS OF BARS

Let us consider thin and uniform bar for longitudinal vibrations as shown. The bar is subjected to axial forces. An element dx of the bar is considered here for analysis.



if u is displacement at a distance x from left end it becomes $u + \frac{du}{dx} dx$ @ $x+dx$.

$$dx + \frac{dy}{dx} dx - dx = \frac{dy}{dx} dx$$

So strain of the element

$$\epsilon = \frac{\frac{dy}{dx} \cdot dx}{dx}$$

$$= \frac{dy}{dx}$$

A = cross sectional area of bar

ρ = density of the material

E = modulus of elasticity of the material

F = Force

Net force

$$\left(F + \frac{dF}{dx} dx \right) - F = m \cdot a$$

$$= dm \times \frac{d^2y}{dt^2}$$

$$\frac{dF}{dx} dx = (\rho dx A) \frac{d^2y}{dt^2}$$

we know

$$\sigma = \frac{F}{A}$$

$$F = \sigma A$$

$$\frac{dF}{dx} = \frac{d\sigma}{dx} A$$

$$\left(\frac{dF}{dx} \right) dx = \left(\frac{d\sigma}{dx} \right) dx A$$

$$\left(\frac{d\sigma}{dx} \right) dx A = \rho dx \times \left(\frac{d^2y}{dt^2} \right)$$

$$\frac{d\sigma}{dx} = \rho \left(\frac{d^2y}{dt^2} \right)$$

TRANSVERSE VIBRATION OF BEAMS

$$\frac{\sigma}{E} = \epsilon$$

$$\sigma = E \epsilon$$

$$\frac{\partial \sigma}{\partial x} = \frac{\partial E}{\partial x} \epsilon$$

$$\left(\frac{\partial \sigma}{\partial x} \right) dx A = \left(\frac{\partial E}{\partial x} \right) dx A \epsilon$$



$$\therefore \frac{\partial E}{\partial x} dx A \epsilon = e dx A \left(\frac{\partial^2 y}{\partial x^2} \right)$$

$$\frac{E}{e} \frac{\partial \epsilon}{\partial x} = \frac{\partial^2 y}{\partial x^2}$$

$$\left(\frac{E}{e} \right) \frac{\partial}{\partial x} \left(\frac{\partial y}{\partial x} \right) = \frac{\partial^2 y}{\partial x^2}$$

$$\frac{E}{e} \left(\frac{\partial^2 y}{\partial x^2} \right) = \frac{\partial^2 y}{\partial t^2}$$

$$\frac{\partial^2 y}{\partial x^2} = \frac{1}{a^2} \frac{\partial^2 y}{\partial t^2}$$

This is 1-D wave equation.

$$-\theta - \left(\theta + \frac{\partial \theta}{\partial x} dx \right) = \theta -$$

$$\left(\frac{\partial^2 \theta}{\partial x^2} \right) (x A dx) = \frac{\partial \theta}{\partial x} -$$

$$a^2 = \frac{E I}{\rho A^2} + \frac{e}{\rho}$$