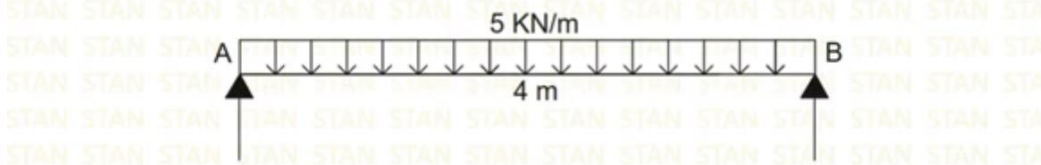




A beam 4m in length is simply supported at its ends and carries uniformly distributed load of 5KN/m over the entire length. Determine strain energy stored in the beam.

$$E = 200 \text{ GPa and } I = 1440 \text{ cm}^4$$



$$I = 1440 \text{ cm}^4 = \frac{1440}{[100]^4} \text{ m}^4 = \frac{1440 \times 10^{-8}}{10^8} \text{ m}^4$$

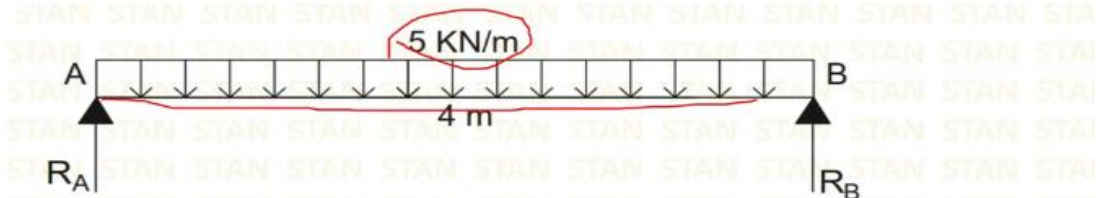
$$E = 200 \text{ GPa} = 200 \times 10^9 \frac{\text{N}}{\text{m}^2} = 200 \times 10^6 \frac{\text{KN}}{\text{m}^2}$$

$$1 \text{ m} = \frac{1}{100} \text{ cm}$$

$$\text{Pascal} = \frac{\text{N}}{\text{m}^2}$$

$$1 \text{ Giga} = 10^9$$

$$1 \text{ Kilo} = 10^3$$



$$\text{Total load in the beam} = 5 \times 4 = 20 \text{ KN}$$

$$R_A = R_B = \frac{20}{2} = 10 \text{ KN}$$

$$\text{Strain Energy due to bending, } U = \frac{1}{2EI} \int_0^L M^2 dx$$

$$\text{Strain energy due to bending, } U = \frac{1}{2EI} \int_0^L M^2 dx$$

$$M = 10x - 5 \times x \times \frac{x}{2}$$



$$M = 10x - 2.5x^2$$

$$U = \frac{1}{2EI} \int_0^4 [10x - 2.5x^2]^2 dx$$

$$U = \frac{1}{2EI} \int_0^4 [10x - 2.5x^2]^2 dx$$

$$= \frac{1}{2EI} \int_0^4 [100x^2 + 6.25x^4 - 50x^3] dx$$

$$= \frac{1}{2EI} \left[ \frac{100x^3}{3} + \frac{6.25x^5}{5} - \frac{50x^4}{4} \right]_0^4$$

$$= \frac{1}{2EI} \left[ \frac{100 \times 4^3}{3} + \frac{6.25 \times 4^5}{5} - \frac{50 \times 4^4}{4} \right]$$

$$= \frac{1}{2EI} [2133.33 + 1280 - 3200]$$

$$= \frac{1}{2EI} [213.33]$$

$$= \frac{1}{2EI} \times 213.33$$

$$= \frac{1}{2 \times 200 \times 10^6 \times 1440 \times 10^{-8}} \times 213.33$$

$$= \frac{213.33}{5760}$$

$$U = 0.03703 \text{ KNm}$$

$$\text{Or } U = 37.03 \text{ Nm}$$

Result:

Strain Energy stored in the beam,  $U = 37.03 \text{ Nm}$  or  $37.03 \text{ j}$



The external diameter of a hollow shaft is twice the internal diameter. It is subjected to a pure torque and it attains a maximum shear stress  $\tau$ . Show that the strain energy stored per unit volume of the shaft is  $\frac{5\tau^2}{16C}$ . Such a shaft is required to transmit

5400 kW at 110 rpm, with uniform torque, the maximum stress not exceeding 84 MN/m<sup>2</sup>. Determine:

- (i) The shaft diameters
- (ii) The energy stored per m<sup>3</sup>.

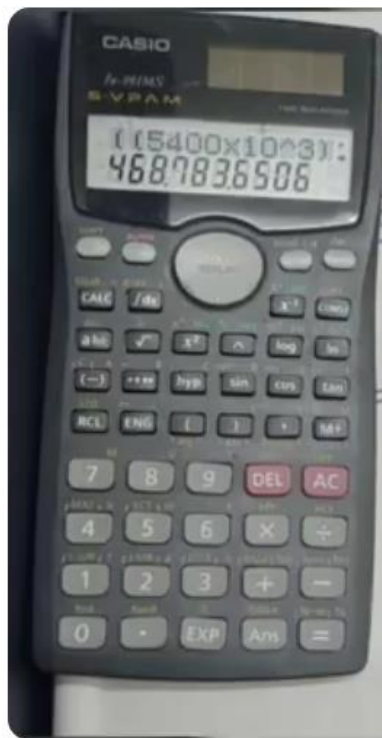
Take:  $C = 90 \text{ GN/m}^2$ .

Sol:- Given data :-

$$\frac{U}{\text{Unit volume}} = \frac{5\tau^2}{16C}$$

$$P = 5400 \text{ kW}, N = 110 \text{ rpm}$$

$$\tau_{\text{max}} = 84 \text{ MN/m}^2$$



$$P = 5400 \text{ kW}, N = 110 \text{ rpm}$$

$$\tau_{\text{max}} = 84 \text{ MN/m}^2 = \frac{84 \times 10^6}{(10^3)^2}$$

$$= \frac{84 \times 10^6}{10^6} = 84 \text{ N/mm}^2$$

$$\tau_{\text{max}} = 84 \text{ N/mm}^2$$

$$P = 5400 \text{ kW} = 5400 \times 10^3 \text{ W}$$

$$1 \text{ kW} = 1000 \text{ W}$$

diameter :-

$$P = \frac{2\pi NT}{60} \quad T = ?$$

$$T = \frac{P \times 60}{2\pi N}$$

$$= \frac{5400 \times 10^3 \times 60}{2 \times \pi \times 110}$$

$$T = 468783 \text{ Nm}$$

$$T = 468783 \times 10^3 \text{ Nmm}$$

$$D = ?$$

By using torsional equation,

$$\frac{T}{J} = \frac{\tau}{R} = \frac{C\theta}{L}$$

$$\frac{T}{J} = \frac{\tau}{R}$$

$$T = J \times \frac{\tau}{R}$$

We know that,  $J = \frac{\pi d^4}{32}$ , if bar is used as Shaft

$$T = \frac{\pi}{32} \times [D^4 - d^4] \times \frac{\tau}{D/2}$$

$\therefore R = \frac{D}{2}$

$$= \frac{\pi}{16} \times \left[ \frac{D^4 - \left(\frac{D^4}{16}\right)}{D} \right] \times \tau$$

$$= \frac{\pi}{16} \left[ \frac{(16D^4 - D^4)}{16} \right] \times \tau$$

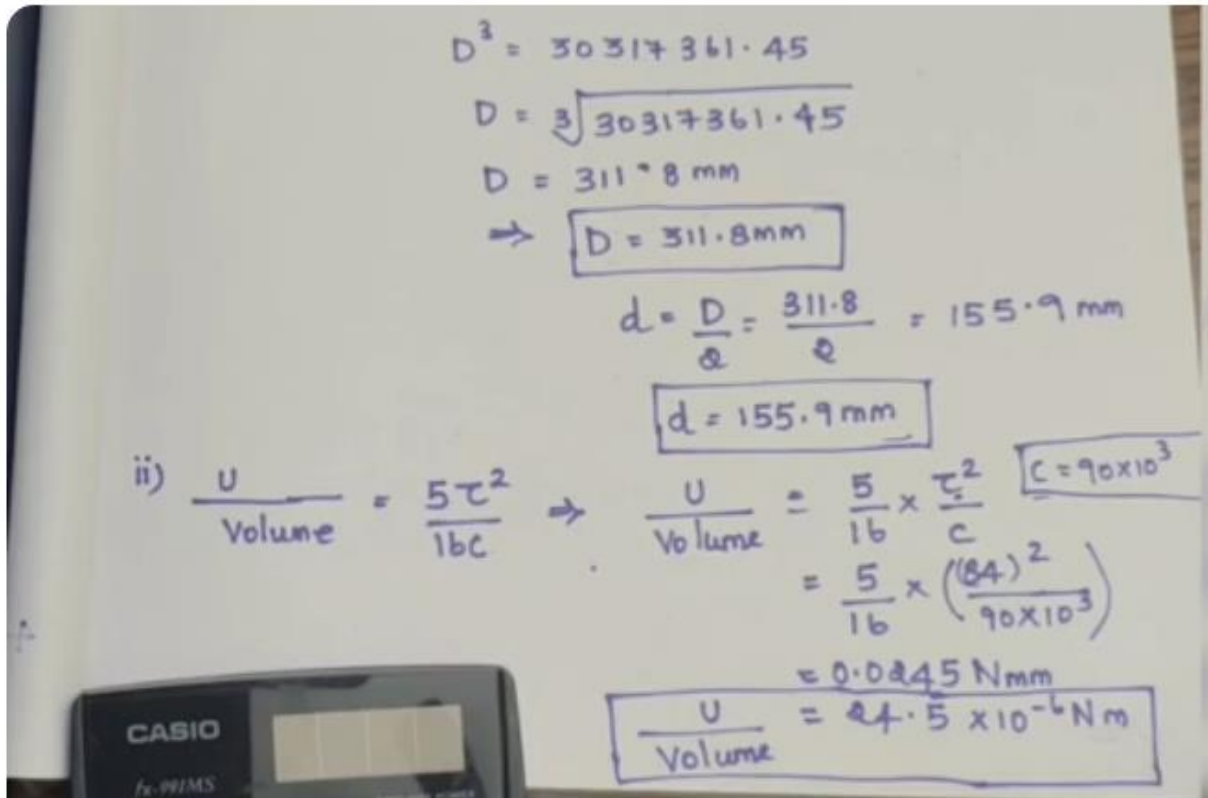
$$= \frac{\pi}{16} \left[ \frac{(15D^4)}{16} \right] \times \tau = \frac{\pi}{16} \left[ \frac{15D^4}{16} \times \frac{1}{D} \right] \times \tau$$

$$= \frac{\pi}{16} \left[ \frac{15D^3}{16} \right] \times \tau = \frac{\pi}{16} \left[ \frac{15D^3}{16} \right] \times \tau$$

$$= \frac{\pi \times 15 \times D^3 \times \tau}{256} \Rightarrow T = \frac{\pi \times 15 \times D^3 \times \tau}{256}$$

$$468783 \times 10^3 = \frac{\pi \times 15 \times D^3 \times \tau_{\max}}{256}$$

$$\Rightarrow D^3 = \frac{468783 \times 10^3 \times 256}{\pi \times 15 \times 84}$$



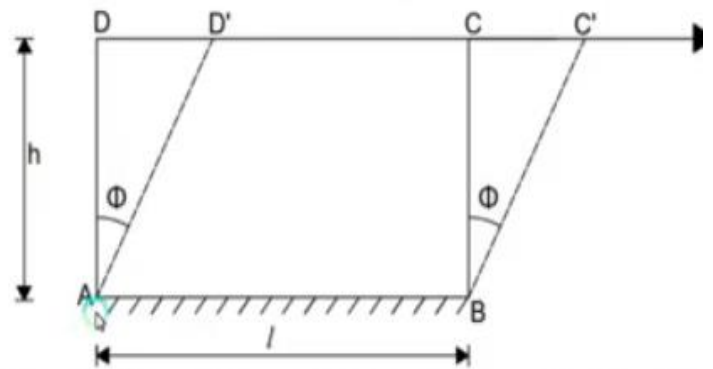
**STRAIN ENERGY DUE TO SHEAR STRESS**

Let

$\tau$  = Shear Stress

$\Phi$  = Shear Strain

S = Shear Force



$$\tau = \frac{\text{Shear Force}}{\text{Area}} = \frac{S}{b \times l}$$

$$S = \tau \times b \times l$$

$$\tan \Phi = \frac{CC'}{CB} = \Phi \text{ (because } \Phi \text{ is very small)}$$

$$CC' = CB \Phi$$

Work done by shear force = Average load  $\times$  distance

$$= \frac{S}{2} \times CC'$$

$$= \frac{1}{2} \times \tau \times b \times l \times CB \times \Phi$$

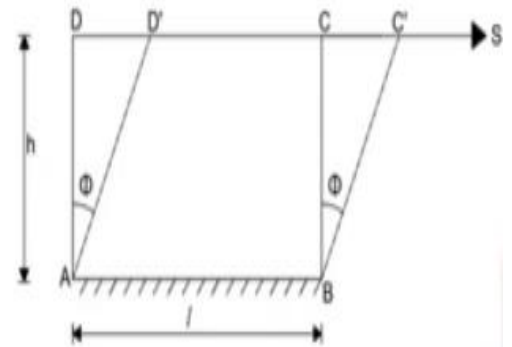
$$= \frac{1}{2} \times \tau \times b \times l \times h \times \frac{\tau}{C}$$

$$= \frac{\tau^2}{2C} \times b \times l \times h$$

$$= \frac{\tau^2}{2C} \times V$$

Strain energy stored = Work done by load

$$\text{Strain energy stored due to shear} = \frac{\tau^2}{2C} \times V$$



$$S = \tau \times b \times l$$

$$CC' = CB \Phi$$

$$\text{Shear Strain } \Phi = \frac{\text{Shear Stress } (\tau)}{\text{Modulus of Rigidity } (C)}$$