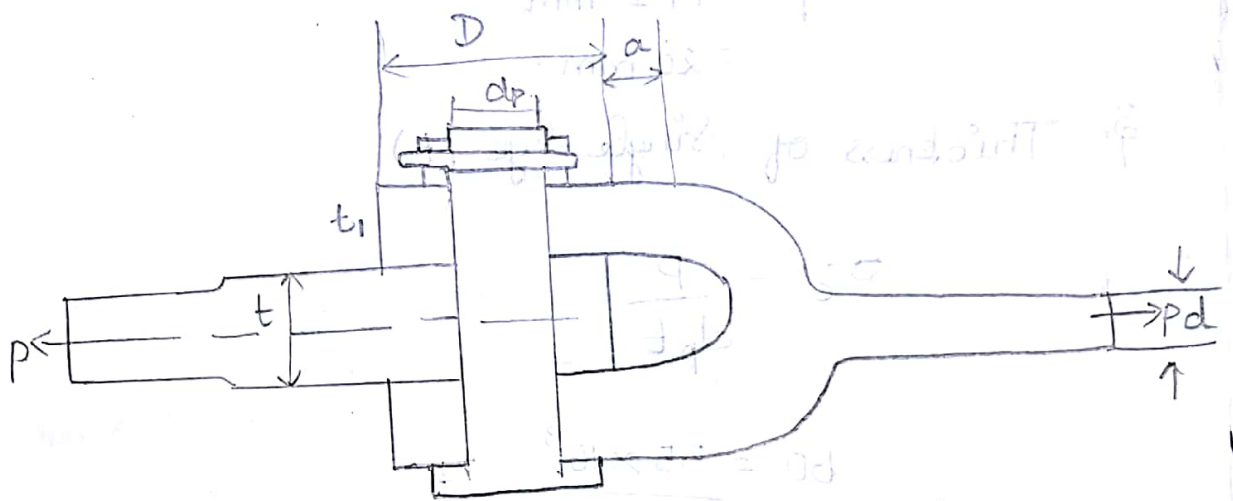


Knuckle Joints

Knuckle joints is used to connect two rods under the action of tensile loads.

- Design a knuckle joint used to connect two rods under the action of tensile load of 55 kN . Allowable crushing stress = 60 MPa . Allowable tensile stress 60 MPa , shear stress 30 MPa .



Diameter of rod.

$$\sigma_t = \frac{P}{(\pi d^2/4)}$$

$$60 = \frac{55 \times 10^3 \times 4}{\pi d^2}$$

$$d^2 = \frac{55 \times 10^3 \times 4}{\pi \times 60}$$

$$d = 34.16 \text{ mm}$$

19, 21, 29, 53, 59

2. Diameter of knuckle pin (d_p).

$$\tau = \frac{P}{A}$$

$$30 = \frac{55 \times 10^3}{\frac{2 \times (\pi d_p)^2}{4}}$$

$$30 = \frac{55 \times 10^3 \times 4}{2 \times \pi^2 \times d_p^2}$$

$$d_p^2 = \frac{55 \times 10^3 \times 4}{2 \times \pi^2 \times 30}$$

$$d_p = 19.2 \text{ mm}$$
$$= 20 \text{ mm}$$

3. Thickness of single eye (t).

$$\sigma_c = \frac{P}{d_p t}$$

$$60 = \frac{55 \times 10^3}{20 \times t}$$

$$t = \frac{55 \times 10^3}{20 \times 60}$$

$$= 45.83 \text{ mm}$$

$$t = 46 \text{ mm}$$

$$t = 1.25 d_p$$

$$= 34.16 \times 1.25$$

$$= 42.7$$

$$= 43 \text{ mm}$$

Thickness = 46 mm.

4. Thickness of fork (t_1).

$$\sigma_c = \frac{P}{2dpt_1}$$

$$60 = \frac{55 \times 10^3}{2 \times 20 \times t_1}$$

$$t_1 = \frac{55 \times 10^3}{2 \times 20 \times 60}$$
$$= 22.91$$

$$t_1 = 23 \text{ mm}$$

$$t_1 = 0.75 \times 34.16$$
$$= 25.62 \text{ mm}$$
$$= 26 \text{ mm}$$

$$t_1 = 26 \text{ mm}$$

5. Outside diameter of eye (D).

$$\sigma_t = \frac{P}{(D-d_p)t}$$

$$60 = \frac{55 \times 10^3}{(D-20)46}$$

$$D-20 = \frac{55 \times 10^3}{60 \times 46}$$

$$D = 40 \text{ mm}$$

$$\tau = \frac{P}{(D-d_p)t}$$

$$(D-20) = \frac{55 \times 10^3}{30 \times 46}$$

$$D = 60 \text{ mm}$$

stress in the fork.

$$\begin{aligned}\sigma_t &= \frac{P}{2(D-d_p)t_f} \\ &= \frac{55 \times 10^3}{2(60-20)26} \\ &= 26.44 \text{ MPas.}\end{aligned}$$

$$\sigma_t < 60 \text{ MPas.}$$

Design is safe.

$$\tau = \frac{P}{2(D-d_p)t_f}$$

$$\begin{aligned}\tau_{30} &= \frac{55 \times 10^3}{2(60-20)26} \\ &= 26.44 \text{ MPas.}\end{aligned}$$

$$\tau < 30 \text{ MPas.}$$

Design is safe.

1. Design a cotter joint to transmit a load of 30 kN in tension (or) compression. Allowable tensile stress is 55 MPas. Crushing stress is 70 MPa and shear stress is 40 MPas.

$$\sigma_t = 55 \text{ MPas}$$

$$\sigma_c = 70 \text{ MPas}$$

$$\tau = 40 \text{ MPas.}$$

1. Diameter of rod

$$\sigma_t = \frac{P}{(\pi d^2/4)}$$

$$55 = \frac{30 \times 10^3 \times 4}{\pi d^2}$$

$$d^2 = \frac{30 \times 10^3 \times 4}{\pi \times 55}$$

$$d = 26 \text{ mm}$$

2. Thickness of cotter

$$t = 0.3d$$

$$= 0.3 \times 26$$

$$t = 8 \text{ mm}$$

3. Diameter of spigot

$$55 = \frac{30 \times 10^3}{\left[\frac{\pi d_1^2}{4} - d_1 \times 8 \right]}$$

$$\frac{\pi d_1^2}{4} - 8d_1 = \frac{30 \times 10^3}{55}$$

$$= 545.45$$

$$\pi d_1^2 - 32d_1 = 2181.818$$

$$d_1 = 32 \text{ mm}$$

$$70 = \frac{30 \times 10^3}{d_1 \times 8}$$

$$d_1 = \frac{30 \times 10^3}{70 \times 8}$$

$$d_1 = 54 \text{ mm}$$

4. Outside diameter of socket (D_1).

$$\sigma_t = \frac{P}{\left[\frac{\pi(D_1^2 - d_1^2)}{4} - (D_1 - d_1)t \right]}$$

$$55 = \frac{30 \times 10^3}{\left[\frac{\pi(D_1^2 - 54^2)}{4} - (D_1 - 54)t \right]}$$

$$\frac{30 \times 10^3}{55} = 0.78(D_1^2 - 54^2) - (D_1 - 54)8$$

$$545.45 = 0.78D_1^2 - 2274.48 - 8D_1 + 432$$

$$0.78D_1^2 - 2274.48 - 8D_1 + 432 - 545.45 = 0.$$

$$0.78D_1^2 - 8D_1 - 2387.93 = 0.$$

$$D_1 = 60.6$$

$$D_1 = 61 \text{ mm.}$$

5. Distance from end of slot to end of pigot.

$$40 = \frac{30 \times 10^3}{2d_1 a}$$

$$a = \frac{30 \times 10^3}{40 \times 2 \times 54}$$

$$a = 7 \text{ mm.}$$

6. Diameter of socket collar (D_2)

$$\sigma_c = \frac{P}{(D_2 - d_1)t}$$

$$70 = \frac{30 \times 10^3}{8(D_2 - 54)}$$

$$D_2 - 54 = \frac{30 \times 10^3}{70 \times 8}$$

$$D_2 = 107.57 \text{ mm}$$

$$D_2 = 108 \text{ mm}$$

7. Thickness of socket collar.

$$t = \frac{P}{2(D_2 - d_1)c}$$

$$c = \frac{30 \times 10^3}{2(108 - 54)40}$$

$$c = 7 \text{ mm}$$

8. Diameter of spigot socket.

$$\sigma_c = \frac{P}{\left[\frac{\pi (d_2^2 - d_1^2)}{4} \right]}$$

$$\pi (d_2^2 - d_1^2) = \frac{30 \times 10^3 \times 4}{70 \times \pi}$$

$$d_2^2 - 54^2 = 545.67$$

$$d_2 = 59 \text{ mm}$$

9. Thickness of spigot collar (t_1)

$$40 = \frac{30 \times 10^3}{\pi \times t_1 \times d_1}$$

$$t_1 = \frac{30 \times 10^3}{\pi \times 40 \times 54}$$

$$t_1 = 5 \text{ mm}$$

10. Width of cotter (b)

$$\tau = \frac{P}{2bt}$$

$$b = \frac{30 \times 10^3}{40 \times 2 \times 8}$$

$$b = 47 \text{ mm}$$

$$\sigma_t = \frac{P(2D_2 - d_1)}{4b^2t}$$

$$b^2 = \frac{30 \times 10^3 (2 \times 108 - 54)}{4 \times 55 \times 8}$$

$$b = 53 \text{ mm}$$

Width of cotter = 53 mm