



SNS COLLEGE OF TECHNOLOGY

AN AUTONOMOUS INSTITUTION



Approved by AICTE New Delhi & Affiliated to Anna University Chennai
Accredited by NBA & Accredited by NAAC with A⁺⁺ Grade Recognized by UGC

DEPARTMENT OF AGRICULTURAL ENGINEERING

COURSE CODE & NAME: 19AGT301 & HEAT POWER ENGINEERING

III YEAR / V SEMESTER

UNIT : IV IC ENGINE PERFORMANCE AND AIR COMPRESSORS

TOPIC 2 :Problems on IC Engine Performance Parameters



Content



- Performance Parameters of IC Engine
- Solved Examples



Example 1 A four-cylinder, two-stroke cycle petrol engine develops 40 kW at 1500 rpm. The mean effective pressure in each cylinder is 7.5 bar. The mechanical efficiency is 85 %. Determine the cylinder bore and piston stroke, if stroke to bore ratio is 1.5. Also determine the fuel consumption if brake thermal efficiency is 30 %. Take calorific value of petrol as 44 MJ/kg.

Solution.

Given : $i = 4$, $n = 1$, $BP = 40$ kw, $N = 1500$ rpm, $P_{mi} = 7.5$ bar,

$\eta_{mech} = 0.85$, $L/D = 1.5$, $CV = 44$ MJ/kg, $(\eta_{th})_b = 0.3$

$$IP = \frac{BP}{\eta_{mech}} = \frac{40}{0.85} = 47.06 \text{ kW}$$

$$IP = \frac{P_{mi} A L N i}{60 \times 10^3 \times \eta}$$

$$47.06 = \frac{7.5 \times 10^5 \times \frac{\pi}{4} d^2 \times 1.5d \times 1500 \times 4}{60 \times 10^3 \times 1}$$

$$d^3 = 5.3261 \times 10^{-4}$$

$$d = 0.081 \text{ m or } 81 \text{ mm}$$

$$L = 1.5d$$

$$= 1.5 \times 81$$

$$= 121.5 \text{ mm}$$

Brake thermal efficiency,

$$(\eta_{th})_b = \frac{BP \times 60}{\dot{m}_f \times CV}$$

$$\dot{m}_f = \frac{40 \times 60}{0.3 \times 44 \times 10^3} = 0.1818 \text{ kg / min}$$



Example 2 A six-cylinder petrol engine operates on the four-stroke cycle. The bore of each cylinder is 70 mm and stroke 100 mm. The clearance volume per cylinder is 67 cm^3 . At a speed of 3960 rpm the fuel consumption is 19.5 kg/h and the torque developed is 140 N.m. Calculate (a) *BP*, (b) *BMEP*, (c) brake thermal efficiency, if LCV of fuel is 44 MJ/kg, and (d) relative efficiency on brake power basis. The engine works on the constant volume cycle basis.

Assume $\gamma = 1.4$ for air.

Solution.

Given : $i = 6$, $n = 2$, $d = 0.07 \text{ m}$, $L = 0.1 \text{ m}$, $V_C = 67 \times 10^{-6} \text{ m}^3$, $N = 3960 \text{ rpm}$,

$LCV = 44 \text{ MJ/kg}$, $\gamma = 1.4$, $\dot{m}_f = 19.5 \text{ kg/h}$, $T = 140 \text{ N.m}$.

$$(a) \quad BP = \frac{2\pi NT}{60 \times 10^3} = \frac{2\pi \times 3960 \times 140}{60 \times 10^3} = 58 \text{ kW}$$

$$(b) \quad BP = \frac{P_{mb} ALNi}{60 \times 10^3 \times n}$$

$$P_{mb} = \frac{58 \times 60 \times 10^3 \times 2}{\frac{\pi}{4} (0.07)^2 \times 0.1 \times 3960 \times 6} = 7.61 \text{ bar}$$



$$(c) (\eta_{th})_b = \frac{BP \times 3600}{\dot{m}_f \times LCV} = \frac{58 \times 3600}{19.5 \times 44 \times 10^3} = 0.244 \text{ or } 24.4 \%$$

$$(d) \text{ Swept volume per cylinder, } V_s = \frac{\pi}{4} d^2 \times L = \frac{\pi}{4} (0.07)^2 \times 0.1 = 3.848 \times 10^{-4} \text{ m}^3$$

$$\text{Compression ratio, } r = \frac{V_s + V_c}{V_c} = \frac{384.8 + 67}{67} = 6.743$$

$$\text{Air standard efficiency, } \eta_a = 1 - \frac{1}{r^{\gamma-1}} = 1 - \frac{1}{(6.743)^{0.4}} = 0.534$$

$$\text{Relative efficiency, } \eta_r = \frac{(\eta_{th})_b}{\eta_a} = \frac{0.244}{0.534} = 0.457 \text{ or } 45.7 \%$$



Example 3 A single-cylinder, four-stroke cycle oil engine is fitted with a rope brake. The diameter of the brake wheel is 600 mm and rope diameter is 26 mm. The dead load on the brake is 200 N and the spring balance reads 30 N. If the engine runs at 450 rpm, what will be the brake power of the engine?



Solution.

Given: $n = 2$, $D_b = 600$ mm, $d_r = 26$ mm, $W = 200$ N, $S = 30$ N, $N = 450$ rpm

$$\begin{aligned}BP &= \frac{(W - S) \pi (D_b + d_r) N}{60 \times 10^3} \\ &= \frac{(200 - 30) \times \pi (0.6 + 0.026) \times 450}{60 \times 10^3} \\ &= 2.507 \text{ kW}\end{aligned}$$



Example 4 A six cylinder, four-stroke, spark ignition engine of bore 10 cm and stroke 12 cm with a compression ratio of 6 is tested at 4800 rpm on a dynamometer of arm 55 cm. During a 10 minute test, the dynamometer reads 450 N and the engine consumes 5 kg of petrol of calorific value 45 MJ/kg. The carburettor receives the air at the rate of 10 kg/min. Calculate :

(a) Brake power, (b) *BSFC*, (c) *BMEP*, (d) *BSAC*, (e) brake thermal efficiency, and (f) air-fuel ration.

Solution. Given : $d = 10$ cm, $L = 12$ cm, $r = 6$, $N = 4800$ rpm, $i = 6$, $n = 2$,

$l = 55$ cm, $t = 10$ min, $W = 450$ N, $m_f = 5$ kg, $CV = 45$ MJ/kg, $\dot{m}_s = 10$ kg / min

(a) Torque,

$$T = Wl = 450 \times 0.55 = 247.5 \text{ N.m}$$

Brake power,

$$BP = \frac{2\pi NT}{60 \times 10^3} = \frac{2\pi \times 4800 \times 247.5}{60 \times 10^3} = 124.41 \text{ kW}$$

$$BP = \frac{P_{mb} ALNi}{60 \times 10^3 \times n}$$

$$124.41 = P_{mb} \times \frac{\pi}{4} (0.1)^2 \times 0.12 \times \frac{4800 \times 6}{60 \times 10^3 \times 2}$$



$$(b) \dot{m}_f = \frac{m_f}{t} = \frac{5 \times 60}{10} = 30 \text{ kg/h}$$

$$BSFC = \frac{\dot{m}_f}{BP} = \frac{30}{124.41} = 0.241 \text{ kg/kWh}$$

$$(c) p_{mb} = 5.5 \text{ bar}$$

$$(d) \dot{m}_a = 10 \times 60 = 600 \text{ kg/kwh}$$

$$BSAC = \frac{600}{124.41} = 4.823 \text{ kg / kWh}$$

$$(e) \eta_{bt} = \frac{BP}{\dot{m}_f \times CV} = \frac{124.41 \times 3600 \times 10^3}{30 \times 45 \times 10^6} = 0.3317 \text{ or } 33.17 \%$$

$$(f) \frac{A}{F} = \frac{\dot{m}_a}{\dot{m}_f} = \frac{600}{30} = 20 : 1$$



Example 5 A six-cylinder, 4-stroke SI engine delivers 400 kW at 2200 rpm. Determine the bore and stroke from the following data :

Compression ratio = 7.6; stroke to bore ratio = 1.25, i.m.c.p. = 100 N/cm², mechanical efficiency = 0.75, calorific value of fuel = 105000 k cal/kg.

What is the brake specific fuel consumption ?

Solution.

Given : $i = 6$, $n = 2$, $BP = 400$ kW, $N = 2200$ rpm, $r = 7.6$, $L/d = 1.25$,

$imep = 100$ N/cm² or 10 Pa, $\eta_{mech} = 0.75$, $CV = 105000 \times 4.184 = 439320$ kJ/kg,

$$IP = \frac{BP}{\eta_{mech}} = \frac{400}{0.75} = 533.33 \text{ kW}$$

$$IP = \frac{p_{mi} \times ALNi}{60 \times 10^3 \times n}$$

$$533.33 = \frac{10 \times 10^5 \times \frac{\pi}{4} d^2 \times 1.25d \times 2200 \times 6}{60 \times 10^3 \times 2}$$

$$d^3 = 4.9386 \times 10^{-3}$$

$$d = 0.1703 \text{ m or } 170.3 \text{ mm}$$

$$L = 1.25 d = 1.25 \times 170.3 = 212.8 \text{ mm}$$



Indicated thermal efficiency,

$$\begin{aligned}(\eta_{th})_i &= 1 - \frac{1}{r^{1-\gamma}} \\ &= 1 - \frac{1}{(7.6)^{0.4}} = 0.5557\end{aligned}$$

$$(\eta_{th})_i = \frac{IP}{\dot{m}_f \times CV}$$

$$\dot{m}_f = \frac{533.33}{0.5557 \times 439320} = 0.00218 \text{ kg / s}$$

$$BSFC = \frac{\dot{m}_f \times 3600}{BP} = \frac{0.00218 \times 3600}{400} = 0.0197 \text{ kg / kWh}$$



Thank You