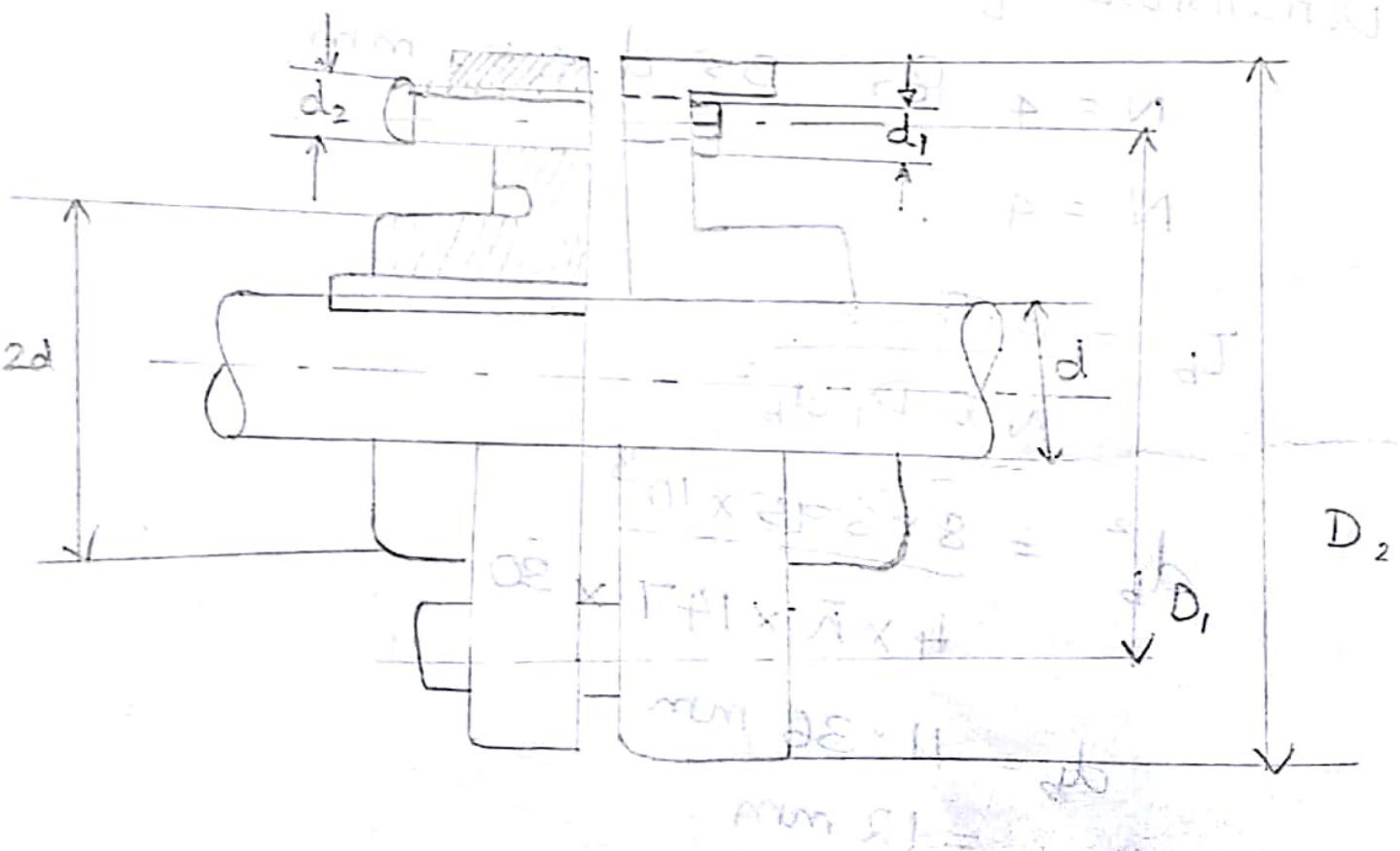


1. Design Bushed pin type flexible flange

Coupling Transmits 30 kW power at 900 rpm from electrical motors to pumps. The maximum torque transmitted is 25% greater than mean torque. Allowable shear stress of shaft & key is 40 MPa. Allowable shear stress of CI flange is 14 MPa. Bearing pressure of pin is 6 MPa.



Given :-

$$P = 30 \text{ kW}$$

$$N = 900 \text{ rpm}$$

$$\sigma_{SK} = 80 \text{ MPa}$$

$$\tau_{SK} = 40 \text{ MPa}$$

$$\tau_{CI} = 14 \text{ MPa}$$

Diameter of shaft :-

$$T_{max} = \frac{16 T_{mean}}{\pi d^3}$$

$$T_{mean} = \frac{60 \times P}{2\pi N}$$
$$= \frac{60 \times 30 \times 10^3}{2\pi \times 900}$$

$$= 318.3 \times 10^3 \text{ N}\cdot\text{mm}$$

$$T_{max} = 1.25 \times T_{mean}$$

$$= 397.8 \times 10^3 \text{ N}\cdot\text{mm}$$

$$40 = \frac{16 \times 397.8 \times 10^3}{\pi d^3}$$

$$d^3 = \frac{16 \times 397.8 \times 10^3}{\pi \times 40}$$

$$d = 37 \text{ mm}$$

Key dimensions -

$$\text{Width of the key} = \frac{d}{4}$$

$$= 9.25 \text{ mm}$$

$$h = \left(\frac{2}{3}\right) w = 6.16 \text{ mm}$$

iii) Length of key:

$$\begin{aligned}l &= 1.5d \\ &= 1.5 \times 37 \\ &= 55.5 \text{ mm}\end{aligned}$$

$$\tau_d = \frac{2T}{dwl}$$

$$40 = \frac{2 \times 397.8 \times 10^3}{37 \times 9.25 \times l}$$

$$l = \frac{2 \times 397.8 \times 10^3}{37 \times 9.25 \times 40}$$

$$l = 58 \text{ mm.}$$

$$\sigma_c = \frac{4T_{\max}}{dlh}$$

$$80 = \frac{4 \times 397.8 \times 10^3}{37 \times 6.16 \times l}$$

$$l = \frac{4 \times 397.8 \times 10^3}{37 \times 6.16 \times 80}$$

$$= 87.26 \text{ mm}$$

$$l = 87 \text{ mm.}$$

iv) Dimension of hub.

i) length of hub $L = l = 87 \text{ mm.}$

Outside diameter $D = 2d$

$$= 2 \times 37$$

$$= 74 \text{ mm.}$$

$$k = d/D = \frac{37}{74} = 0.5$$

$$\tau_h = \frac{16T}{\pi D^3 (1-k^4)}$$

$$= \frac{16 \times 397.8 \times 10^3}{\pi \times 74^3 (1-0.5^4)}$$

$$= \frac{6364800}{1193483.19}$$

$$= 5.334 \text{ MPa} < 14 \text{ MPa}$$

Design is safe.

Dimensions of flange.

$$t_f = 0.5d = 0.5 \times 37$$

$$= 18.5 \text{ mm}$$

$$t_p = 0.25d$$

$$= 9.25 \text{ mm}$$

$$\tau_f = \frac{2T}{\pi D^2 t_f}$$

$$= \frac{2 \times 397.8 \times 10^3}{\pi \times 74^2 \times 18.5}$$

$$= 2.49 \text{ N/mm}^2 < 14 \text{ MPa}$$

Design is safe.

5. Dimension of pin.

i) Number of pins.

$$N=4$$

ii) Nominal diameter of pin.

$$\begin{aligned}d_b &= \frac{0.5d}{\sqrt{N}} \\&= \frac{0.5 \times 37}{\sqrt{4}} \\&= 9.25 \text{ mm.}\end{aligned}$$

$$\begin{aligned}d_1 &= d_b + 4 \\&= 13.25 \text{ mm.}\end{aligned}$$

$$\begin{aligned}d_2 &= d_1 + 16 \text{ mm} \\&= 13.25 + 16 \\&= 29.25 \text{ mm.}\end{aligned}$$

Diameter of bolt circle

$$\begin{aligned}D_1 &= D + d_2 + 16 \\&= 74 + 29.25 + 16 \\&= 119.25 \text{ mm.}\end{aligned}$$

Length of bush in flange.

$$P_b = \frac{2T}{N D_1 d_2 l_b}$$

$$l_b = \frac{2T}{N D_1 d_2 P_b}$$

$$\begin{aligned}&= \frac{2 \times 397.8 \times 10^3}{4 \times 119.25 \times 29.25 \times 6}\end{aligned}$$

$$l_b = 9.5 \text{ mm} .$$