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#### **DEPARTMENT OF AGRICULTURE ENGINEERING**

#### **19AGB401 - SOLAR AND WIND ENERGY**

IV – YEAR VII SEMESTER

**UNIT 1 – SOLAR ENERGY RADIATION AND SOLAR THERMAL COLLECTORS** 

19AGB401 SOLAR AND WIND ENERGY.---Ms.J.Hemalatha,AP/AGE

# Outline





- What are flat plate collectors?
- Types of flat plate collectors
- Applications of flat plate collectors
- Thermal analysis of flat plate collectors

### What Are Flat Plate Collectors?



- A flat plate collector is a heat exchanger that uses solar irradiation to heat a working fluid.
- The working fluid is usually liquid or air.
- The collector is a black surface that is placed at a convenient path of the sun.
- In flat plate collectors there is no optical concentration of sunlight and they are generally stationary.
- The outlet temperature capability is below 100 °C





## What Are Flat Plate Collectors?





- A typical flat plate collector is a metal box with a glass or plastic cover (called glazing) on top and a dark-colored absorber plate on the bottom.
- The sides and bottom of the collector are usually insulated to minimize heat loss.







# ABSORBER PLATE

- The plate is usually made of copper, steel, or plastic.
- The surface is covered with a black material of high absorptance.
- A selective coating can be used to maximize the absorptance of solar energy and minimizes the radiation emitted by plate.







# FLOW PASSAGES

- The flow passages carry the working fluid through the collector.
- If the working fluid is a liquid, the flow passage is usually a tube that is attached to, or is a part of absorber plate.
- If the working fluid is air, the flow passages can have different configurations.







# COVER PLATE (GLAZING)

- To reduce convective and radiative heat losses from the absorber, one or two transparent covers (glazing) are generally placed above the absorber plate.
- They usually be made from glass or plastic.







## INSULATION

These are some materials such as fiberglass and they are placed at the back and sides of the collector to reduce heat losses.





## ENCLOSURE

- A box that encloses the collector to:
  - Hold all the components together
  - Protect them from weather
  - Facilitate installation on a roof or appropriate frame.



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## WATER SYSTEMS



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## **AIR SYSTEMS**





# Outline





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## DOMESTIC HOT WATER







## WATER HEATING









## **SPACE HEATING**



# Outline





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## **Energy** Absorbed by a Flat Plate Collector





- The irradiation incident on a collector ( $G_t$ ) is not all absorbed.
- Once the irradiation penetrates the glass cover, part of it is absorbed by the collector, but another part is reflected back diffusely to the glass cover.
  - The glass cover then reflects diffusely to the absorber, and so on.



### **Energy** Absorbed by a Flat Plate Collector





- The net energy absorbed by the collector can be expressed in terms of a quantity called ( $\tau \alpha$ ).
  - Theoretically, the net energy absorbed by the collector per unit area is:
  - $S = G_t (\tau \alpha)_{av}$
  - Where  $(\tau \alpha)_{av}$  is the average value of  $(\tau \alpha)$



Energy Gain of a Flat Plate Collector



The useful energy gain of a flat plate collector is given by:

 $Q_{\rm u} = S \times A_{\rm c} - Q_{\rm loss}$ 

- $Q_{\rm loss}$  can be due to energy loss through:
- Top of the collector
- Bottom of collector
- Edges of collector

 $Q_{\text{loss}}$  can be expressed as:  $Q_{\text{loss}} = U_{\text{L}} A_{\text{c}} (T_{\text{p}} - T_{\text{a}})$ where,

- $T_p$ : mean temperature of the absorber plate
- *T*<sub>a</sub>: ambient temperature
- U<sub>L</sub>: overall heat transfer coefficient based on collector area

### Energy Loss of a Flat Plate Collector



 $U_{\rm L}$  consists of  $U_{\rm t}$  (top),  $U_{\rm b}$  (bottom), and  $U_{\rm e}$  (edges)



Energy Loss of a Flat Plate Collector

 $U_{\rm t}$  involves the calculation of:

- $h_{c,p-g}$ : convective heat transfer coefficient between plate and glazing
  - *h*<sub>r,p-g</sub>: radiative heat transfer coefficient between plate and glazing
  - $h_{c,g-a}$ : convective heat transfer coefficient between glazing and ambient air
  - $h_{\rm r,g-a}$ : radiative heat transfer coefficient between glazing and ambient air





Calculations of all the components of U<sub>4</sub> is complicated.

Energy Loss of a Flat Plate Collector



\*A relatively simple alternative formula can be used:

$$\begin{split} U_t &= \frac{1}{\frac{N_g}{\left[\frac{C}{T_p} - T_a\right]^{0.33} + \frac{1}{h_w}}} \\ &+ \frac{\sigma(T_p^2 + T_a^2)(T_p + T_a)}{\frac{1}{\varepsilon_p + 0.05N_g(1 - \varepsilon_p)} + \frac{2N_g + f - 1}{\varepsilon_g} - N_g} \\ f &= (1 - 0.04h_w + 0.0005h_w^2)(1 + 0.091N_g) \\ C &= 365.9 \left(1 - 0.00883 \ \alpha + 0.0001298 \ \alpha^2\right) \\ h_w &= \frac{8.6V^{0.6}}{L^{0.4}} \text{ igage of solar and wind energy.--Ms_J.Hemaladha,AP/AGE} \end{split}$$

## Energy Loss of a Flat Plate Collector



- $N_g$  = number of glass covers
- $T_{p}$  = absorber plate temperature
- $T_a$  = ambient temperature
- $\sigma$  = Stefan-Boltzmann constant
- $\varepsilon_p$  = emittance of absorber plate
- $\varepsilon_g$  = emittance of glazing
- $\alpha$  = Tilt angle
- V = Wind velocity
- L = Collector length



Energy Loss of a Flat Plate Collector





•  $U_{\rm b}$  can be found from:

$$U_b = \frac{1}{\frac{t_b}{k_b} + \frac{1}{h_{c,b-a}}}$$

## Where,

- $t_b$  = thickness of back insulation (m).
- $k_b$  = conductivity of back insulation (W/m-K).

 $h_{c, b-a}$  = convection heat loss coefficient from back to ambient (W/m<sup>2</sup>-K).

Energy Loss of a Flat Plate Collector



 $U_{\rm e}$  can be found from:

$$U_e = \frac{1}{\frac{t_e}{k_e} + \frac{1}{h_{c,e-a}}}$$

## Where,

- = thickness of edge insulation (m). t<sub>e</sub> k<sub>e</sub>
  - = conductivity of edge insulation (W/m-K).

 $h_{c, e-a}$  = convection heat loss coefficient from edge to ambient (W/m<sup>2</sup>-K).

Energy Gain of a Flat Plate Collector



The useful energy gain of a flat plate collector is given by:

$$Q_{\rm u} = S \times A_{\rm c} - Q_{\rm loss}$$

Expanding all terms,

$$Q_u = A_c [G_t(\tau \alpha) - U_L(T_p - T_a)]$$

The useful energy gained by the collector is transferred completely to the working fluid. Therefore,

$$Q_u = A_c [G_t(\tau \alpha) - U_L(T_p - T_a)] = \dot{m}c_p [T_o - T_i]$$

Where,

- *T<sub>i</sub>*: fluid inlet temperature
- T<sub>o</sub>: fluid outlet temperature



 $F_R$ 



$$Q_u = A_c [G_t(\tau \alpha) - U_L(T_p - T_a)] = \dot{m}c_p [T_o - T_i]$$

- Calculating  $T_p$  accurately is difficult.
- It is more convenient to express Q<sub>u</sub> in terms of the fluid temperatures.
- A useful definition is the heat removal factor ( $F_R$ ):

Actual output

Output for plate temperature = Fluid inlet temperature

#### 🛺 Heat Removal Factor





• By using the heat removal factor ( $F_R$ ), the useful energy gain equation  $Q_u \underset{c}{\text{becAm}} [G_t(\tau \alpha) - U_L(T_i - T_a)]$ 

- $F_R$  depends on many factors, and it can be found analytically.
- $F_R$  can also be found experimentally.

## Collector Efficiency





- Collector efficiency  $(\eta)$  is defined as the:
- $\eta$  = useful energy gain / irradiation incident on the collector

$$\eta = \frac{Q_u}{G_t A_c}$$

This equation can be expressed in terms of  $F_R$ :

$$\eta = F_R \left[ (\tau \alpha) - \frac{U_L (T_i - T_a)}{G_t} \right]$$





• The efficiency equation can be rearranged as follows:

$$\eta = F_R(r\alpha) - F_R U_L \frac{T - T_a}{G_t}$$

- If changes in  $F_R$  and  $U_L$  are small, the equation above represents a straight line, where:
  - $(T_i T_a) / G_t$  is the independent variable
  - $\eta$  is the dependent variable
  - $F_R(\tau \alpha)$  is the intercept
  - (-  $F_R U_L$ ) is the slope

**Collector Efficiency** 







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