

## CANTILEVER:

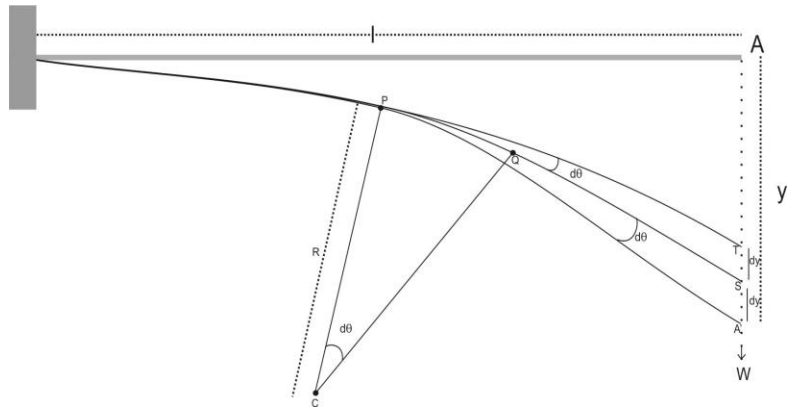
A Cantilever is a beam fixed horizontally at one end and loaded at other end.

### Depression of a cantilever loaded at its ends:

#### Theory:

The cantilever OA is fixed at O, its length is  $l$  and 'W' be the weight loaded at other end. Due to load it moves to OA'.

Let us consider an element PQ of the beam of length  $dx$ , at a distance  $x$  from fixed end. 'C' be the centre of curvature and R be the radius of curvature.



Due to the load (W) applied at free end, an external couple is created between A and Q, arm of couple is  $(l - x)$ .

$$\text{External bending movement} = W \times (l - x) \quad \text{-----(1)}$$

$$\text{Internal bending movement} = \frac{YI}{R} \quad \text{-----(2)}$$

Under equilibrium condition,

$$\text{External bending movement} = \text{Internal bending movement}$$

$$\therefore R = \frac{YI}{W(l - x)} \quad \text{----- (3)}$$

From the figure arc length

$$PQ = R d\theta = dx$$

$$d\theta = \frac{dx}{R} \quad \text{-----(4)}$$

On substituting R Value

$$d\theta = \frac{dx}{YI} W(l - x) \quad \text{-----(5)}$$

From  $\Delta QAS$

$$\sin\theta = \frac{dy}{l - x}$$

$$dy = d\theta(l - x) \text{ ----- (6)}$$

On sub (5) in (6) we get,

$$dy = \frac{W(l - x)^2}{YI} dx$$

∴ Total depression is by integrating the above within the limit 0 to l.

$$\therefore y = \frac{W}{YI} \int_0^l (l - x)^2 dx$$

On solving we get,

$$y = \frac{W}{YI} \cdot \frac{l^3}{3}$$

This equation gives the depression of the cantilever.

Special Cases:

(i) Rectangular cross section. For,

$$I = \frac{bd^3}{12}$$

$$\text{Depression produced } y = \frac{4Wl^3}{Ybd^3}$$

(ii) Circular cross section, For  $I = \frac{\pi r^4}{4}$

$$\text{Depression Produced } y = \frac{4Wl^3}{3\pi r^4 Y}$$

'r' is the radius of the circular cross section.

## Young's Modulus(Y):

It is defined as the ratio between the longitudinal stress to the longitudinal strain, within the elastic limits.

$$\text{Young's Modulus (Y)} = \frac{\text{longitudinal stress}}{\text{longitudinal strain}}$$

## **Uniform Bending**

Consider a beam supported by two knife edges A and B. Length between A and B is 'l'. Let equal weights (W), be added to either end of the beam C and D.

Let the distance CA and BD = a. Due to load applied the elevation 'x' produced from F to E. Let W be the reaction produced at the points A and B which acts vertically upwards.

From the fig

The external bending moment about P, written as

$$M_p = Wa$$

We know the internal bending moment =  $\frac{YI_g}{R}$

On comparing (1) and (2)

$$Wa = \frac{YI_g}{R}$$

Here it is found that the elevation 'x' forms an arc of the circle of radius R

From  $\Delta AFO$

$$OA^2 = AF^2 + FO^2$$

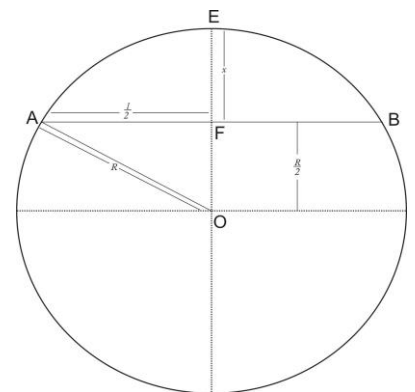
Since  $OF = FE$ , therefore we can write  $OA^2 = AF^2 + FE^2$

$$\text{(or)} \quad AF^2 = OA^2 - FE^2$$

$$AF^2 = FE \left[ \frac{OA^2}{FE} - FE \right]$$

Here,  $AF = \frac{l}{2}$ ,  $FE = x = \frac{R}{2}$ ;  $OA = R$

$$\left( \frac{l}{2} \right)^2 = x \left[ \frac{R^2}{R/2} - x \right]$$



$$\left(\frac{l^2}{4}\right) = 2xR - x^2$$

If elevation  $x$  is small

$$\left(\frac{l^2}{4}\right) = 2xR$$

$$x = \frac{l^2}{8R} \quad (\text{or}) \quad R = \frac{l^2}{8x}$$

$$(\text{or}) \quad Wa = \frac{YI_g}{l^2/8x} \quad (\text{or}) \quad Wa = \frac{8YI_g x}{l^2}$$

On Rearranging

The elevation of point 'E' above 'A' is  $x = \frac{Wal^2}{8YI_g}$