



**SNS COLLEGE OF TECHNOLOGY**

**19MCB204 -  
SOLIDMECHANICS**

**UNIT- IV DEFLECTION OF BEAMS AND BUCKLING OF  
COLUMNS**

**Macaulay Method-Problems**



## *Macaulay Method-Problems*

A steel girder of uniform section, 14 meters long is simply supported at its ends. It carries concentrated loads of 90 KN and 60 KN at two points 3 meters and 4.5 meters from the two ends respectively. Calculate: (i) The deflection of the girder at the points under the two loads.(ii)The maximum deflection. Take: $I = 64 \times 10^{-4} \text{ m}^4$  and  $E = 210 \times 10^6 \text{ KN/m}^2$ .



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Solution:-

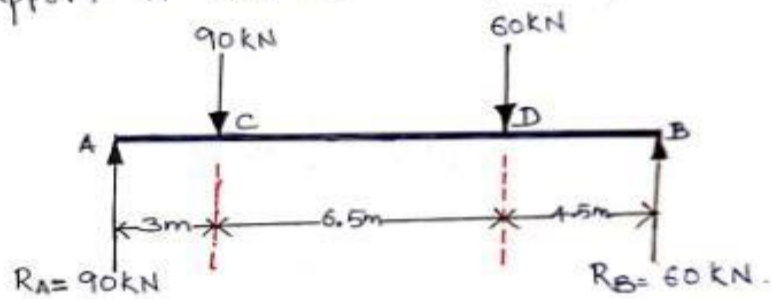
Span of steel girder,

$$l = 14.0 \text{ m.}$$

Moment of Inertia,  $I = 64 \times 10^{-4} \text{ m}^4.$

Young's Modulus,  $E = 210 \times 10^6 \text{ kN/m}^2.$

Let  $R_A$  and  $R_B$  be the reactions at support A and B respectively.



Taking moments about A, we get.

$$R_B \times 14 = 90 \times 3 + 60 \times 9.5 = 840.$$

$$R_B = 60 \text{ kN.}$$

$$R_A + R_B = 90 + 60 = 150 \text{ kN}$$

$$R_A = 150 - 60 = 90 \text{ kN}$$

$$R_A = 90 \text{ kN.}$$



## Macaulay Method-Problems

Consider any section  $xx$  at a distance  $x$  from end  $A$ , following Macaulay's method, the bending moment is given by,

$$M_x = EI \frac{d^2y}{dx^2} = 90x - 90(x-3) - 60(x-9.5) \rightarrow \textcircled{1}$$

Integrating, we get.

$$EI \frac{dy}{dx} = 45x^2 + C_1 - 45(x-3)^2 - 30(x-9.5)^2 \rightarrow \textcircled{2}$$

Integrating again, we get.

$$EI y = 15x^3 + C_1 x + C_2 - 15(x-3)^3 - 10(x-9.5)^3 \rightarrow \textcircled{3}$$

When,  $x=0, y=0$ .

$$\therefore C_2 = 0.$$

When,  $x=14m, y=0$ .

$$0 = 15 \times (14)^3 + C_1 \times 14 - 15(14-3)^3 - 10(14-9.5)^3$$

$$= 41160 + 14C_1 - 19965 - 911.25 = 14C_1 - 20283.75$$

$$C_1 = -1448.84$$



## Macaulay Method-Problems

Hence the deflection at any section is given by.

$$EI\delta = 15x^3 - 1448.84x - 15(x-3)^3 - 10(x-9.5)^3$$

→ Deflection Equation.

i)  $\delta_c$  and  $\delta_D$  :-

Deflection at C,  $\delta_c$ .

Putting  $x = 3m$  in the deflection equation, we get.

$$EI\delta_c = 15 \times 3^3 - 1448.84 \times 3 = 405 - 4346.52$$

$$EI\delta_c = -3941.52$$

$$\delta_c = \frac{-3941.52}{EI} = \frac{3941.52}{210 \times 10^6 \times 64 \times 10^{-4}}$$

$$\delta_c = -0.00291 \text{ m} = -2.93 \text{ mm}$$

(or)

Downward deflection of C.

$$\delta_c = -2.93 \text{ mm}$$



## Macaulay Method-Problems

Deflection at  $D, y_D$ : Putting  $x = 9.5$  mm in the downward deflection equation, we get.

$$EI y_D = 15 \times 9.5^3 - 1448.84 \times 9.5 - 15(9.5-3)^3$$

$$\Rightarrow 12860.6 - 13764 - 419.45 = -5022.8$$

$$y_D = \frac{-5022.8}{EI}$$
$$= \frac{-5022.8}{210 \times 10^6 \times 64 \times 10^{-4}}$$

$$y_D = -0.00373 \text{ m (or)} -3.73 \text{ mm}$$

Downward deflection of

$$D = 3.73 \text{ mm}$$

ii) Maximum deflection,  $y_{\max}$ :-

Let us assume that the deflection will be maximum at a section between C and D. Equating the slope at the section to zero, we get.



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***Macaulay Method-Problems***





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$$EI \frac{dy}{dx} = 45x^2 - 1448.84 - 45(x-3)^2 = 0.$$

Or,

$$45x^2 - 1448.84 - 45(x^2 - 6x + 9) = 0.$$

Or,

$$45x^2 - 1448.84 - 45x^2 + 270x - 405 = 0.$$

Or,

$$270x = +1853.84.$$

$$x = 6.87m$$

Putting this value of  $x$  in the deflection equation, we get.

$$EI y_{max} = 15 \times 6.87^3 - 1448.84 \times 6.87 - 15(6.87 - 3)^3.$$

$$= 4863.6 - 9953.5 - 869.4 = -5959.3.$$

$$y_{max} = -\frac{5959.3}{EI}$$

$$= -\frac{5959.3}{210 \times 10^6 \times 64 \times 10^{-4}}$$

$$y_{max} = 0.0043m \text{ (or) } -4.43mm$$

Downward deflection,  $y_{max} = 4.43mm$ .