



#### **SNS COLLEGE OF TECHNOLOGY**

# 19MCB204 – SOLIDMECHANICS

# UNIT- IV DEFLECTION OF BEAMS AND BUCKLING OF COLUMNS

**Equivalent length of a column - Problems** 





#### TABLE: 1

Case	End Conditions.	Equivalent Buckling land,	
1.	Both ends hinged (or) Pin jointed (or) wounded (or) were.	(	$\frac{\mathcal{I}^2 E \mathcal{I}}{\mathcal{L}^2} = \frac{\mathcal{I}^2 E \mathcal{I}}{\mathcal{L}^2}$
2.	One and fixed, Other and tree	20	$\frac{T^2ET}{(e^2)} = \frac{T^2ET}{4(2)}$
3.	One end fixed, Other end Pin Jointed.	V2 1	$\frac{2ET}{(e^2)} = \frac{2X^2ET}{(e^2)}$
A	Both ends fixed (ov) encastered.	2 1	2 EI = 472 EI

## SNS CULLEGE OF TECHNOLOGY





# **Equivalent length of a column**





## **SNS COLLEGE OF TECHNOLOGY**





## **Equivalent length of a column - Problems**

#### Tolution:

Dut side diameter of the column,

.D=50mm = 0.05m

Inside diameter of the Column, d.

Area of the Column,

$$A = \frac{\pi}{4} (D^2 - d^2) = \frac{\pi}{4} (0.05^2 - d^2)$$

Moment of Inertia of the Column,

$$I = \frac{\pi}{64} (D^4 - d^4) = \frac{\pi}{64} (0.05^4 - d^4)$$

Also, Critical load = 2x sate (oad (Given).

= 2x 13.6 = 27.2 KN.

End Conditions: Pin ended.

Using the relation, 
$$P_{\text{Enler}} = \frac{T^2 E I}{(e^2)}$$
, We get.

$$27.2 \times 10^{3} = \frac{\pi^{2} \times 70 \times 10^{9} \times \frac{\pi}{64} (0.05^{4} - d^{4})}{1.8^{2}}$$

$$(0.054-d4) = \frac{27.2\times10^{3}\times1.8^{2}\times64}{T^{2}\times70\times10^{9}\times7} = 2.6\times10^{6}$$

Inside diameter, d= 43.7mm





A slender pin ended aluminum column 1.8 m long and of circular cross-section is to have an outside diameter of 50 mm. Calculate the necessary internal diameter to prevent failure by buckling if the actual load applied is 13.6 kN and the critical load applied is twice the actual load. Take E for aluminum as 70 GN/m2.





#### **SNS COLLEGE OF TECHNOLOGY**

### **Equivalent length of a column - Problems**

A bar of length 4m when used as a simply supported beam and subjected to a u.d.l. of 30 kN/m over the whole span, deflects 15 mm at the centre. Determine the crippling loads when it is used as a column with following end conditions:a) Both ends pin-jointed;b) One end fixed and other end hinged;c) Both ends fixed.





# Solution:

# Given Data:

Length of the bar 1=4m

uniformly distributed load, W=30KN/m.

Deflection, S= 15mm=0.015m.

We know that,  $8 = \frac{5W14}{284 ET}$ 

 $0.015 = \frac{5 \times (30 \times 10^{3}) \times 4^{4}}{384 \text{ EI}}$ 

 $EI = \frac{5 \times (30 \times 10^{3}) \times 4^{4}}{0.015 \times 384} = 6.66 \times 10^{6} \text{Nm}^{2}$ 

i) 
$$P_{\text{Eulev}} = \frac{\overline{\Lambda^2 \text{EI}}}{(e^2)^2} ((e^2 (-4m)).$$
  
=  $\overline{\Lambda^2 \times 6.66 \times 10^6} = 4108 \text{ kN}$ 

11) 
$$P_{Enlev} = \frac{T^2 E I}{\binom{2}{\epsilon^2}} \qquad \left[ \left( e^{-\frac{1}{\sqrt{2}}} + \frac{4}{\sqrt{2}} + \frac{2.83 \text{ m}}{\sqrt{2}} \right) \right]$$
  
=  $\frac{T^2 E I}{2.83^2} = \frac{T^2 \times 6.66 \times 10^6}{2.83^2} = \frac{8207 \text{ kN}}{2.83^2}$ 

ii) 
$$P_{\text{Eulev}} = \frac{T^2 E T}{l_e^2} \left[ \left( e = \frac{l}{2} = \frac{4}{2} = 2^m \right) \right]$$
  
=  $\frac{T^2 E T}{2^2} = \frac{T^2 \times 6.66 \times 10^6}{2^2} = \frac{16432 \text{ kN}}{2^2}$