

# SNS COLLEGE OF TECHNOLOGY

Coimbatore-35

An Autonomous Institution

Accredited by NBA – AICTE and Accredited by NAAC – UGC with 'A++' Grade  
Approved by AICTE, New Delhi & Affiliated to Anna University, Chennai



## DEPARTMENT OF AEROSPACE ENGINEERING

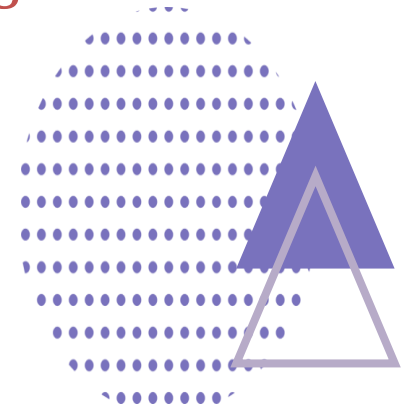
### 16AE315-THEORY OF VIBRATIONS

III YEAR VI SEM

#### UNIT II – SINGLE DEGREE OF FREEDOM SYSTEMS

#### TOPIC 4 – VIBRATION MEASURES

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# Harmonic Disturbances (Spring mass system)

The amplitude of the forced vibration is given by

$$x = \frac{F_0}{\sqrt{(s - m\omega^2)^2 + (c\omega)^2}} \sin(\omega t - \varphi)$$

$F_0$  is the excited force and  $\varphi$  is the phase lag



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The maximum amplitude of the forced vibration is given by



# Harmonic Disturbances (Spring mass system)

## 1. Phase Lag $\phi$

As the displacement takes place after applying force, the displacement vector lags the force vector by some angle  $\phi$ . This angle is known as phase lag.

Mathematically,

$$\phi = \tan^{-1} \left( \frac{2\epsilon r}{1 - r^2} \right)$$

where  $r$  is the frequency ratio



# Harmonic Disturbances (Spring mass system)

## 2. Magnification factor or Dynamic Magnifier

The ratio of maximum displacement of the forced vibration ( $X_{max}$ ) to the static deflection ( $X_o$ ) due to static force and it is denoted by M.F.

$$M.F = \frac{X_{max}}{X_o}$$

$$M.F = \frac{1}{\sqrt{(1 - r^2)^2 + (2\zeta r)^2}}$$



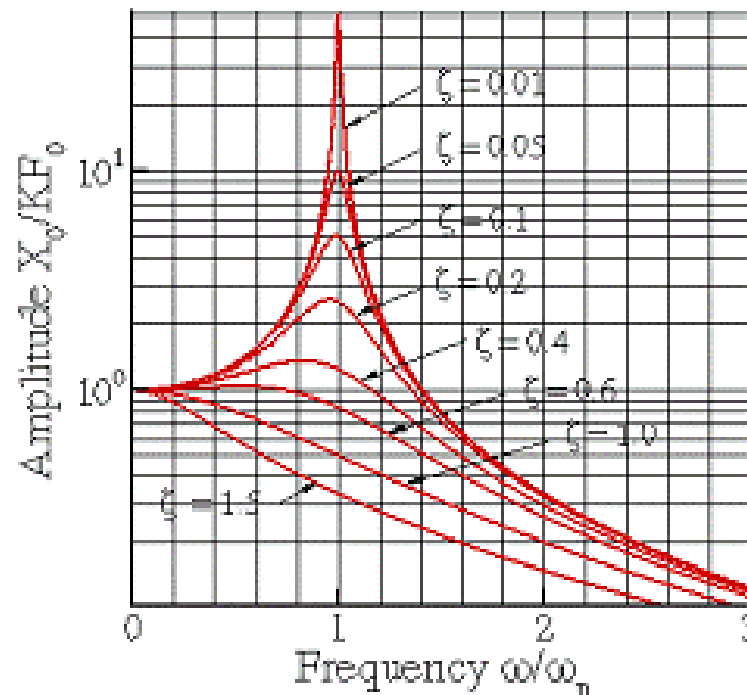
# QUESTIONS RELATED TO ABOVE SLIDES



# Harmonic Disturbances – Characteristics Curve

## 1. Frequency response curve

A curve drawn between magnification factor and frequency ratio is known as frequency response curve.

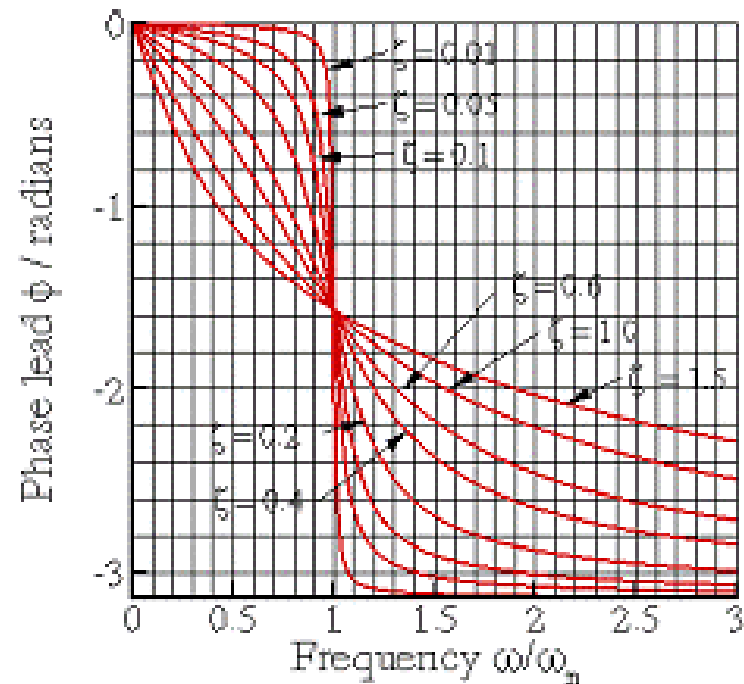




# Harmonic Disturbances – Characteristics Curve

## 2. Phase - frequency response curve

A curve drawn between phase angle and frequency ratio is known as phase frequency response curve.







# Disturbance caused by unbalance

Almost in all rotating and reciprocating machinery like an electric motor, a turbine, an IC engine, etc have some amount of unbalanced force left in them even after correcting their unbalance on precision balancing machines. This unbalance produces the exciting force in a machine.



# Disturbance caused by unbalance

The amplitude of the forced vibration is given by

$$x = \frac{m_u \omega^2 e}{\sqrt{(s - m\omega^2)^2 + (c\omega)^2}} \cos(\omega t - \varphi)$$

$m_u$  - unbalanced mass, and  $e$  - eccentricity

The maximum amplitude of the forced vibration is given by

$$x_{max} = \frac{m_u \omega^2 e}{\sqrt{\sqrt{(s - m\omega^2)^2 + (c\omega)^2}}}$$

$$\frac{x_{max}}{\left(\frac{m_u \cdot e}{m}\right)} = \frac{r^2}{\sqrt{(1 - r^2)^2 + (2\epsilon r)^2}}$$



# Disturbance caused by unbalance

Phase Lag  $\phi$

As the displacement takes place after applying force, the displacement vector lags the force vector by some angle  $\phi$ . This angle is known as phase lag.

Mathematically,

$$\phi = \tan^{-1} \left( \frac{2er}{1 - r^2} \right)$$

where  $r$  is the frequency ratio



# Forced Vibration due to the excitation of the support (Support Motion)

- Absolute amplitude

The amplitude of the forced vibration is given by

$$x_{max} = \frac{Y \sqrt{s^2 + (c\omega)^2}}{\sqrt{\sqrt{(s - m\omega^2)^2 + (c\omega)^2}}}$$
$$\frac{x_{max}}{Y} = \frac{\sqrt{1 + (2\epsilon r)^2}}{\sqrt{(1 - r^2)^2 + (2\epsilon r)^2}}$$

where Y is the amplitude

$$\text{Phase angle, } \varphi = \tan^{-1} \left( \frac{2\epsilon r}{1 - r^2} \right)$$



# Forced Vibration due to the excitation of the support (Support Motion)

- Relative amplitude

In many cases, it is useful to know the response of the system relative to a moving system.

The steady state relative amplitude of the forced vibration is given by

$$\frac{z}{Y} = \frac{r^2}{\sqrt{(1-r^2)^2 + (2\epsilon r)^2}}$$

where  $z$  is the relative motion of the mass with respect to support

$$\text{Phase angle, } \phi = \tan^{-1} \left( \frac{2\epsilon r}{1-r^2} \right)$$