

Macaulay's method

### **Macaulay's method**

Macaulay's method is suitable

- i) When the beam is subjected to an eccentric point load.
- ii) When the beam is subjected to a number of concentrated loads.

### **Changes of Macaulay's method**

- 1.Brackets are to be integrated as a whole.
- 2.Constants of integration are written after the first term.
- 3.The section, for which BM equation is to be written, should be taken in the last part of the beam.

### **Method of Singularity functions**

In Macaulay's method a single equation is formed for all loading on a beam, the equation is constructed in such a way that the constant of Integration apply to all portions of the beam. This method is also called method of singularity functions.

### **The rules observed using Macaulay's method**

Always take origin on the extreme of the beam.

Take left clockwise moment as negative and left counter clockwise moment as positive.

Take a section in the least segment of the beam and take moment from the left.

If the beam carries a UDL, extend it upto the extreme right and superimpose a UDL equal and opposite to that, which has been added while extending the given UDL.

### Uses of Macaulay's Method

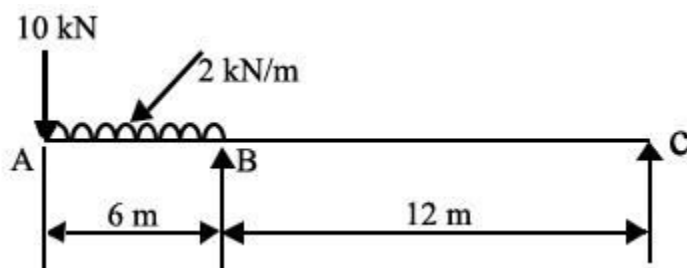
When the problem of deflection in beams are a bit tedious and laborious.

When the beam is carrying several point loads.

It is used to find deflection where BM is discontinuous.

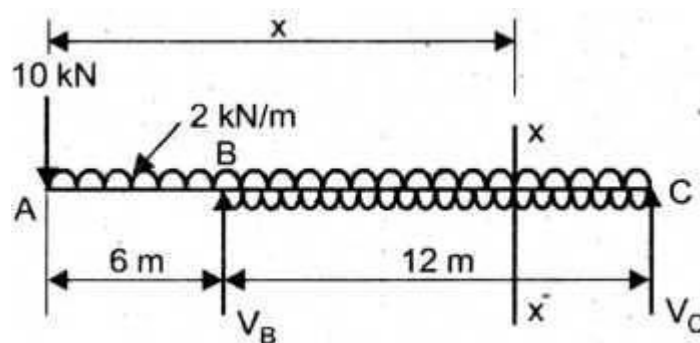
Problems

### Let us solve deflection of Macaulay's method



### **Solution:**

Use Macaulay's method,



Extend the UDL up to the right support C and apply upward UDL from B to C of same magnitude (2 kN/m) to compensate.

Support Reactions

Applying  $\sum V = 0$  ( $\uparrow +$ )

$$V_B + V_C = 10 + (2 \times 6) = 22 \text{ KN}$$

Applying  $\sum M_c = 0$  ( $\curvearrowright +$ )

$$(V_B \times 12) = (2 \times 6 \times 15) + (10 \times 18)$$

Solving,  $V_B = 30 \text{ KN}$  ( $\uparrow$ )

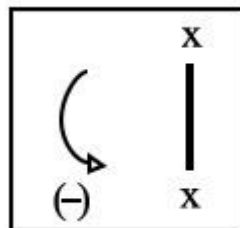
$$\therefore V_C = 22 - 30 = -8 \text{ KN} = 8 \text{ KN} (\downarrow)$$

Consider a section XX at a distance of  $x$  from the end A

$$M_x = V_B(x - 6) - 10x + 2(x - 6) \left[ \frac{(x - 6)}{2} - 2 \left( x \times \frac{x}{2} \right) \right]$$

$$= V_B(x - 6) + 2 \frac{(x - 6)^2}{2} - 10x - \frac{2x^2}{2}$$

$$= V_B(x - 6) + (x - 6)^2 - 10x - x^2$$



or  $EI \frac{d^2 y}{dx^2} = M_x$

$$= V_B(x - 6) + (x - 6)^2 - 10x - x^2$$

Integrating once,

$$EI \frac{dy}{dx} = \frac{30}{2}(x-6)^2 + \frac{(x-6)^3}{3} - \frac{10x^2}{2} - \frac{x^3}{3} + C_1$$

Slope equation ( $\because V_B = 30$ )

Integrating again,

$$EIy = \frac{15(x-6)^3}{3} - \frac{(x-6)^4}{12} - \frac{10x^3}{6} - \frac{x^4}{12} + C_1x + C_2$$

$$= 5(x-6)^3 - \frac{(x-6)^4}{12} - \frac{10x^3}{6} - \frac{x^4}{12} + C_1x + C_2$$

or  $EI y$

To Find the constants

at  $x = 6; y = 0$

Substituting these values in the above equation,

$$0 = \frac{-10 \times 6^3}{6} - \frac{6^4}{12} + (C_1 \times 6) + C_2$$

$$0 = -360 - 108 + 6C_1 + C_2$$

$$\text{or } 468 = 6C_1 + C_2 \dots (i)$$

at  $x = 18 \text{ m}; y = 0$

Substituting these values in the above equation,

$$0 = 5(18-6)^3 - \frac{(18-6)^4}{12} - \frac{10}{6}(18)^3 - \frac{18^4}{12} + (C_1 \times 18) + C_2$$

$$= 8640 + 1728 - 9720 - 8748 + 18C_1 + C_2$$

$$= -8100 + 18C_1 + C_2$$

$$\text{or } 8100 = 18C_1 + C_2 \dots (\text{ii})$$

Solve equations (i) and (ii)

$$468 = 6C_1 + C_2$$

$$8100 = 18C_1 + C_2$$

$$-7632 = -12C_1$$

$$\therefore C_1 = 636$$

Substituting,  $C_2 = -3348$

$$\therefore EI y = 5(x - 6)^3 + \frac{(x - 6)^4}{12} - \frac{10x^3}{6} - \frac{x^4}{12} + 636x - 3348$$

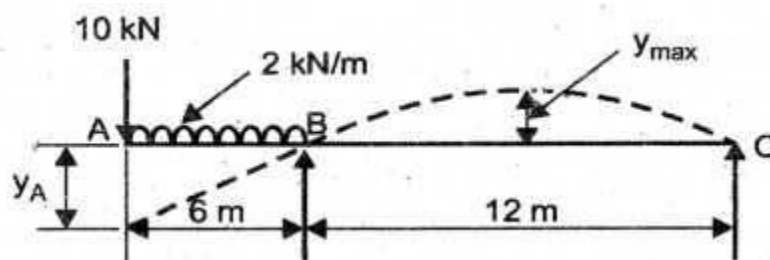
i) Maximum Downward deflection

Occurs at Free end A. Hence substitute  $x = 0$  in the above equation,

$$EI y_A = -3348$$

$$\text{or } y_A = \frac{-3348}{EI} = \frac{-3348 \times 10^{12}}{40000 \times 10^9} = 83.7 \text{ mm}$$

ii) Maximum upward deflection



The deflected shape of beam is shown in Figure above. The maximum upward deflection occurs in the region BC, i.e.,  $6 < x < 12$ .

Location of Maximum deflection (i.e., zero slope)

Using slope equation,

$$EI \frac{dy}{dx} = 15(x - 6)^2 + \frac{(x - 6)^3}{3} - 5x^2 - \frac{x^3}{3} + C_1$$

$$0 = 15(x - 6)^2 + \frac{(x - 6)^3}{3} - 5x^2 - \frac{x^3}{3} + 636$$

i.e.,

Solve this equation by trial and error,

at  $x = 12\text{m}$ , LHS = 0, RHS = -48

at  $x = 13\text{m}$ , LHS = 0, RHS = -91.67

at  $x = 12.5\text{m}$ , LHS = 0, RHS = -70.96

at  $x = 11.5\text{m}$ , LHS = 0, RHS = -23

at  $x = 11\text{m}$ , LHS = 0, RHS = 4.01

$\therefore x$  lies in between 11 m and 11.5 m

Take  $x$  is approximately 11.25 m

Substituting  $x = 11.25$  in deflection equation to find maximum positive deflection,

$$= 5(x - 6)^3 + \frac{(x - 6)^4}{12} - \frac{10x^3}{6} - \frac{x^4}{12} + 636x - 3348$$

$\therefore Ely$

$$\begin{aligned}
\therefore EI y_{\max} &= 5(11.25 - 6)^3 + \frac{(11.25 - 6)^4}{12} + \frac{10 \times 11.25^3}{6} \\
&\quad - \frac{11.25^4}{12} + (636 \times 11.25) - 3348 \\
&= 723.51 + 63.3 + 2.373 - 1334 + 7155 - 3348 \\
&= 3262
\end{aligned}$$

$$\begin{aligned}
y_{\max} &= \frac{3262}{EI} \\
&= \frac{3262 \times 10^{12}}{40000 \times 10^9} = 81.55 \text{ mm.}
\end{aligned}$$

**We shall discuss the problem in the beam shown below, determine the slope at the left end C and the deflection at 1m from the left end. Take  $EI = 0.63 \text{ MN m}^2$**

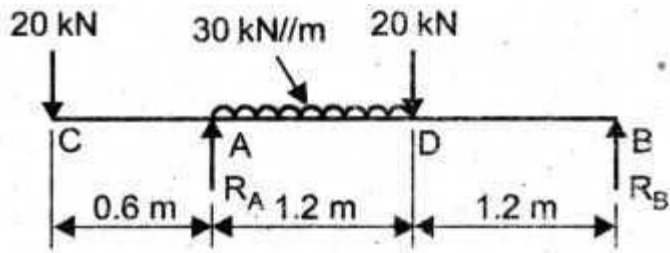
***Given:***

$$EI = 0.65 \text{ MNm}^2$$

$$= 0.65 \times 10^6 \times (1000)^2$$

$$= 0.65 \times 10^{12} \text{ Nmm}^2.$$

***Solution:***



Find  $\theta_C$  and  $y$  at 1 m from the left end.

Support Reactions

Applying  $\sum V = 0$

$$R_A + R_B = (30 \times 1.2) + 20 + 20 = 76 \text{ kN} \dots (1)$$

Applying  $\sum M_A = 0$

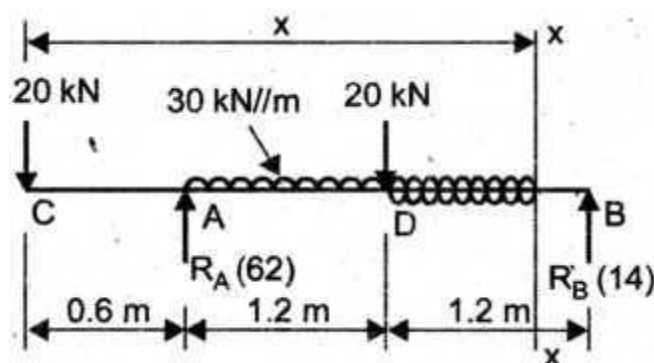
$$(R_B \times 2.4) + (20 \times 0.6) = (20 \times 1.2) + \left( 30 \times 1.2 \times \frac{1.2}{2} \right)$$

$$2.4 R_B + 12 = 24 + 21.6$$

Solving  $R_B = 14 \text{ kN}$

Substituting  $R_B$  in equation (1),

$$R_A = 62 \text{ kN}$$



**Macaulay's method**



Consider a section XX at a distance of  $x$  from C, in the region DB. Extend UDL 30 KN/m upto the section XX and apply the counter UDL in the opposite direction (i.e., upward) from D to the section as shown in figure.

**Bending Moment at section XX**

$$(BM)_x = M = (62 \times (x - 0.6)) - 20(x - 1.8) - 20x$$

$$- \left\{ 30(x - 0.6) \times \frac{(x - 0.6)}{2} \right\} + \left\{ 30(x - 1.8) \times \frac{(x - 1.8)}{2} \right\}$$

$$EI \frac{d^2y}{dx^2} = M = 62(x - 0.6) - 20(x - 1.8) - 20x - 15(x - 0.6)^2 + 15(x - 1.8)^2.$$

Integrating we get

$$EI \frac{dy}{dx} = \frac{62(x - 0.6)^2}{2} - \frac{20(x - 1.8)^2}{2} - \frac{20x^2}{2}$$

$$- \frac{15(x - 0.6)^3}{3} + \frac{15(x - 1.8)^3}{3} + C_1$$

$$= 31(x - 0.6)^2 - 10(x - 1.8)^2 - 10x^2 - 5(x - 0.6)^3 + 5(x - 1.8)^3 + C_1$$

Again Integrating

$$EI y = \frac{31(x - 0.6)^3}{3} - \frac{10(x - 1.8)^3}{3} - \frac{10x^3}{3}$$

$$- \frac{5(x - 0.6)^4}{4} + \frac{5(x - 1.8)^4}{4} + C_1x + C_2$$

**To find  $C_1$  and  $C_2$**

Apply the boundary conditions

at  $x = 0.6\text{m}, y = 0$

Substituting in above equation,

$$0 = \left( \frac{-10}{3} \times (0.6)^3 \right) + (C_1 \times 0.6) + C_2$$

$$0.6C_1 + C_2 = 0.72 \dots (A)$$

at  $x = 3\text{m}, y = 0$

substituting we get

$$0 = \frac{31}{3}(3 - 0.6)^3 - \frac{10}{3}(3 - 1.8)^3 - \frac{10}{3}(3)^3$$
$$- \frac{5}{4}(3 - 0.6)^4 + \frac{5}{4}(3 - 1.8)^4 + (3C_1) + C_2$$

$$0 = 142.85 - 5.76 - 90 - 41.47 + 2.59 + 3C_1 + C_2$$

$$0 = 8.21 + 3C_1 + C_2$$

$$\therefore 3C_1 + C_2 = -8.21 \dots (B)$$

Solve the equations (A) & (B)  $0.6C_1 + C_2 = 0.72$

$$\begin{array}{r} 3C_1 + C_2 = -8.21 \\ \hline -2.4C_1 = 8.93 \end{array}$$

$$\therefore C_1 = -3.72$$

Substituting  $C_1$  in equation (A),

$$C_2 = 2.95$$

Final Slope Equation is

$$EI \frac{dy}{dx} = 31(x - 0.6)^2 - 10(x - 1.8)^2 - 10x^2 - 5(x - 0.6)^3 + 5(x - 1.8)^3 - 3.72$$

Final Deflection Equation is

$$EI y = \frac{31}{3}(x - 0.6)^3 - \frac{10}{3}(x - 1.8)^3 - \frac{10}{3}x^3 - \frac{5}{4}(x - 0.6)^4 + \frac{5}{4}(x - 1.8)^4 - 3.72x + 2.95$$

To find slope at left end C

Substitute  $x = 0$  in Final slope equation

$$EI \theta_c = -3.72 \text{ (Note : Negative terms are neglected)}$$

$$\therefore \theta_c = \frac{-3.72}{EI} = \frac{-3.72 \times 10^9}{0.65 \times 10^{12}}$$

$$= 0.00572 \text{ rad. (anticlockwise slope)}$$

$$= 0.00572 \times \frac{180}{\pi} = 0.327^\circ$$

Deflection at 1 m from left end

Substitute  $x = 1$  m in final deflection equation

$$\therefore EIy = \frac{31}{3}(1-0.6)^3 - \frac{10}{3}(1)^3 - \frac{5}{4}(1-0.6)^4 - (3.72 \times 1) + 2.95$$

**(Note: Negative terms are neglected)**

$$= 0.66 - 3.33 - 0.032 - 3.72 + 2.95$$

$$= -3.472$$

$$\therefore y = \frac{-3.472}{EI} = \frac{-3.472 \times 10^{12}}{0.65 \times 10^{12}} = 5.34 \text{ mm}$$

Problems

Let us solve using Macaulay's method, a beam of length 6 m is simply supported at its ends and carries two point loads of 48 KN and 40 KN at a distance of 1 m and 3 m respectively from the left support. Find i) deflection under each load, ii) maximum deflection and iii) the point at which maximum deflection occurs.

Given  $E = 2 \times 10^5 \text{ N/mm}^2$  and  $I = 85 \times 10^6 \text{ mm}^4$ .

$$E = 2 \times 10^5 \text{ N/mm}^2;$$

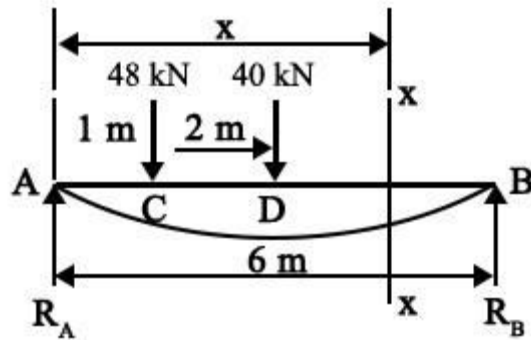
$$I = 85 \times 10^6 \text{ mm}^4$$

$$y_c = ?$$

$$y_d = ?$$

$$y_{\max} = ?$$

**Solution:**



Let  $R_A$  and  $R_B$  be the support Reactions.

Applying  $\sum V = 0 (\uparrow = \downarrow)$

$$R_A + R_B = 48 + 40 = 88 \text{ KN} \dots (i)$$

Applying  $\sum M_A = 0 (\curvearrowright = \curvearrowleft)$

$$(48 \times 1) + (40 \times 3) - (R_B \times 6) = 0$$

Solving,  $R_B = 28 \text{ KN}$

Substituting  $R_B = 28$  in equation (1),

$$R_A = 60 \text{ KN}$$

### ***Applying Macaulay's method to find deflection***

Consider a section XX at a distance of  $x$  from the left support a such that it covers all the loads i.e., consider the section XX in DB region as shown in figure.

$$EI \frac{d^2 y}{dx^2} = M$$

Using the relation,

$$\text{Now, } (BM)_x = M = 60x - 48(x - 1) - 40(x - 3)$$

$$\therefore EI \frac{d^2y}{dx^2} = 60x - 48(x-1) - 40(x-3)$$

Integrating the above equation,

$$EI \frac{dy}{dx} = \frac{60x^2}{2} - \frac{48(x-1)^2}{2} - \frac{40(x-3)^2}{2} + C_1 \dots\dots(i)$$

$$= 30x^2 - 24(x-1)^2 - 20(x-3)^2 \text{ (Slope equation)}$$

Again integrating the equation (i), we get

$$EI y = \frac{30x^3}{3} - \frac{24(x-1)^3}{3} - \frac{20(x-3)^3}{3} + C_1x + C_2 \dots(ii)$$

(Deflection equation)

To Find the constants

Apply the Boundary conditions

i) at  $x = 0$ ;  $y = 0$  and

ii) at  $x = 6m$ ;  $y = 0$

Substituting  $x = 0$  in equation (ii)  $0 = C_2$

Substituting  $x = 6$  in equation (ii)

$$0 = \frac{30}{3}(6)^3 - \frac{24}{3}(6-1)^3 - \frac{20}{3}(6-3)^3 + 6C_1$$

$$0 = \left( \frac{30}{3} \times (6)^3 \right) - \left( \frac{24}{3} \times 5^3 \right) - \left( \frac{20}{3} \times 3^3 \right)$$

$$= 2160 - 1000 - 180 + 6C_1$$

$$0 = 980 + 6C_1$$

$$C_1 = -163.33$$

Now, substituting the values of  $C_1$  and  $C_2$  in equation (ii) we get final deflection equation.

$$\therefore EI y = 10x^3 - 8(x-1)^3 - \frac{20}{3}(x-3)^3 - 163.33x$$

i) Deflection under point loads

Deflection under 48 KN ( i.e.,  $y_c$  )

Substitute  $x = 1\text{m}$  in Final deflection equation

$$EI y = 10x^3 - 8\cancel{(x-1)^3} - \frac{20}{3}\cancel{(x-3)^3} - 163.3x$$

$$\text{or } EI y = 10x^3 - 163.3x$$

**(Note: substituting  $x = 1$  if the value is negative within the brackets  $(x - 1)$  and  $(x - 3)$ , these terms are neglected)**

$$\therefore EI y_c = 10 \times 1^3 - (163.3 \times 1)$$

$$= -153.3$$

$$\therefore y_c = \frac{-153.3}{EI} = \frac{-153.3 \times 10^{12}}{2 \times 10^5 \times 85 \times 10^6}$$

$$= -9.017 \text{ mm}$$

**(Note that in  $EI y_c = -153.3$ , load in KN and distance is in meter so unit of  $-153.3$  is  $\text{KN m}^3$ . Converting  $\text{KN.m}^3$  into  $\text{Nmm}^3$  multiply by  $10^3$  ( $(10^3)^3 = 10^{12}$ ).**

### ***Deflection under 40 KN ( i.e., $y_D$ )***

Substitute  $x = 3$  m in final deflection equation.

$$EI y = 10x^3 - 8(x-1)^3 - \frac{20}{3} (x-3)^3 - 163.3x$$

$$EI y_D = 10x^3 - 8(3-1)^3 - (163.3 \times 3)$$

$$= 270 - 64 - 489.9$$

$$= -283.9$$

$$\therefore y_D = \frac{-283.9}{EI} = \frac{-283.9 \times 10^{12}}{2 \times 10^5 \times 85 \times 10^6}$$

$$= -16.7 \text{ mm}$$

### ***To find point of maximum deflection***

Deflection is maximum where the slope is zero. Referring the deflected shape of beam, maximum deflection will occur in the region CD i.e.,  $1 < x < 3$ .

Consider the slope equation,

$$\text{i.e., } EI \frac{dy}{dx} = 30x^2 - 24(x-1)^2 - 20(x-3)^2 - 163.3$$

Since  $1 < x < 3$ , neglect  $(x-3)$  term

$$\therefore EI \frac{dy}{dx} = 30x^2 - 24(x-1)^2 - 163.3$$

$$0 = 30x^2 - 24(x^2 + 1 - 2x) - 163.3$$



$$= 30x^2 - 24x^2 - 24 + 48x - 163.3$$

$$0 = 6x^2 + 48x - 187.3$$

$$\begin{aligned}\therefore x &= \frac{-48 \pm \sqrt{(48)^2 - (4 \times 6 \times (-187.3))}}{2 \times 6} \\ &= \frac{-48 \pm 82.45}{12}\end{aligned}$$

Neglecting Negative value,  $x = 2.87$  m

$\therefore$  Maximum deflection occurs at a distance of 2.87 m from the left support.

### ***To find maximum deflection***

Substitute the value of  $x = 2.87$ m in final deflection equation.

$$EI y = 10x^3 - 8(x-1)^3 - \frac{20}{3}(x-3)^3 - 163.3x$$

$x = 2.87$ , so neglecting negative term

$$EI y = 10x^3 - 8(x-1)^3 - 163.3x$$

$$EI y_{\max} = 10 \times (2.87)^3 - 8(2.87-1)^3 - (163.3 \times 2.87)$$

$$= 236.4 - 52.31 - 468.67$$

$$= -284.58$$

$$y_{\max} = \frac{-284.58}{EI} = \frac{-284.58 \times 10^{12}}{2 \times 10^5 \times 85 \times 10^6}$$

$$= -16.74 \text{ mm}$$

