



ENGINEERING MECHANICS

QP QA 14 marks questions

Solved Questions

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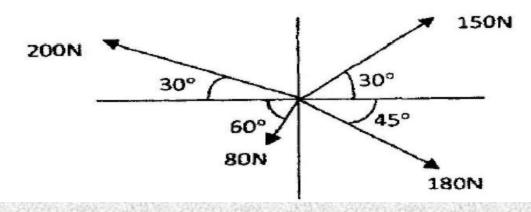
16ME 102 /Engineering Mechanics

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Determine the resultant of the concurrent force system shown in the following Figure. (AU JUN'10, DEC'10, DEC'12)



Solution:

Resolving forces horizontally,

$$\Sigma H = 150\cos 30 - 200\cos 30 - 80\cos 60 + 180\cos 45$$

$$= 130 - 173.2 - 40 + 127.28$$

$$\Sigma H = 44.08 \text{ N}$$

Resolving forces vertically

$$\Sigma V$$
 = 150 sin30 + 200 sin30 - 80 sin60 - 180 sin45
= 75 + 100 - 69.28 - 127.28

$$= -21.56 \text{ N}$$





Magnitude of Resultant

R =
$$\sqrt{\Sigma H^2 + \Sigma V^2}$$

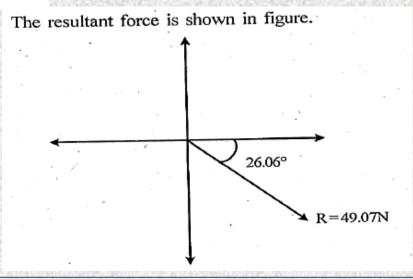
= $\sqrt{(44.08)^2 + (21.56)^2}$

R = 49.07 N

Direction of resultant is

$$\tan \theta = \frac{\Sigma V}{\Sigma H}$$

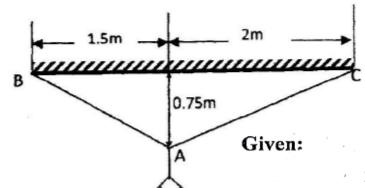
$$\theta = \tan^{-1} \left(\frac{21.56}{44.08} \right) = 26.06 \text{ (IV Quadrant)}$$







The following figure shows a 10 kg lamp supported by two cables AB and AC. Find the tension in each cable. (AU JUN'10, DEC'10, DEC'12)



mass of lamp = 10 kgweight, w = $10 \times 9.81 = 98.1 \text{ N}$

To find:

- 1) Tension in cable AB, $T_{A} = ?$
- 2) Tension in cable AC, $T_{AC} = ?$

Solution:

From the given figure,

In angle ABD,
$$\tan \theta_1 = \frac{0.75}{1.5} = 26.56$$

In angle ACD,
$$\tan \theta_2 = \frac{0.75}{2} = 20.55$$





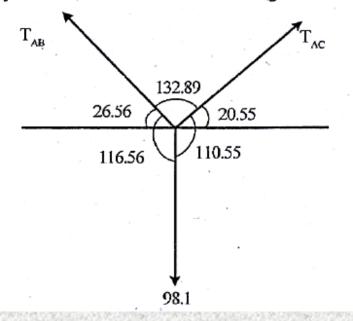
Solution:

From the given figure,

In angle ABD,
$$\tan \theta_1 = \frac{0.75}{1.5} = 26.56$$

In angle ACD,
$$\tan \theta_2 = \frac{0.75}{2} = 20.55$$

The system of forces is shown in figure.



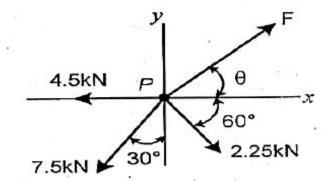
Applying Lami's theorem

$$\frac{T_{AC}}{\sin 116.56} = \frac{T_{AB}}{\sin 110.55} = \frac{98.1}{\sin 132.8}$$





Determine the magnitude and angle θ and F so that particle shown in figure, is in Equilibrium (AU MAY'11, JUN'12)



Given:

Inclination of force, $F = \theta$, with x-axis
Inclination of force 4.5KN = 0 with x-axis
Inclination of force 7.5KN with x-axis = $(90-30) = 60^{\circ}$ Inclination of force 2.25KN with x-axis = 60°

To Find:

$$F = ?$$
; $\theta = ?$





Solution:

It is given that the particle 'P' is in equilibrium. Hence Hence, $\Sigma H = 0$ and $\Sigma V = 0$.

Resolving forces horizontally,

$$\Sigma H = F \cos \theta - 4.5 \cos 0^{\circ} - 7.5 \cos 60^{\circ} + 2.25 \cos 60^{\circ}$$

 $0 = F \cos \theta - 4.5 - 3.75 + 1.125$

$$F \cos \theta = 7.125$$

.... (1)

Resolving forces vertically,

$$\Sigma V = F \sin \theta + 4.5 \sin 0^{\circ} - 7.5 \sin 60^{\circ} - 2.25 \sin 60^{\circ}$$

$$0 = F \sin \theta + 0 - 6.495 - 1.948$$

$$F \sin \theta = 8.443$$

.... (2)





$$\frac{(2)}{(1)} \Rightarrow \frac{F \sin \theta}{F \cos \theta} = \frac{8.443}{7.125}$$

$$\tan \theta = 1.185$$

$$\theta = 49.84^{\circ}$$
(Ans)
$$Put \theta = 49.84^{\circ} \text{ in eqn. (1), we get}$$

$$F \cos(49.84) = 7.125$$

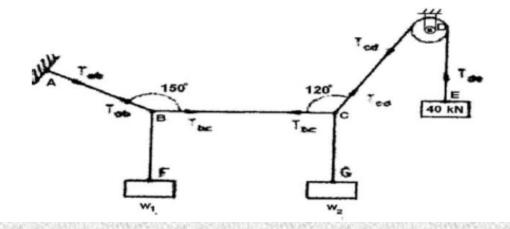
F = 11 KN

(Ans)





ABCDE is a light string whose end A is fixed. The weights W₁ and W₂ are attached to the string at B & C and the string passes round a small smooth wheel at D carrying a weight 40KN at the free end E. In the position of equilibrium, BC is horizontal and AB and CD make angles 150° and 120° with horizontal. (AU DEC'12)



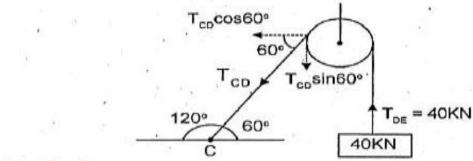




Find (i) the tensions in AB, BC and DE of the given string (ii) magnitudes of W₁ and W₂.

Solution:

Let us consider the pulley first the various forces acting are shown below:



At point D,

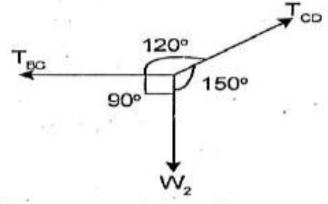
$$T_{DE} = 40 \text{ kN}$$
 $\Sigma Fy = 0$
 $-T_{CD} \sin 60^{\circ} T_{DE} = 0$
 $-0.866 T_{CD} + 40 = 0$
 $T_{CD} = 46.18 \text{ kN}$

(Ans)





Now consider joint C, the various forces acting at C is shown below



Applying Lami's Theorem at joint C,

$$\frac{T_{CD}}{\sin 90^{\circ}} = \frac{T_{BC}}{\sin 150^{\circ}} = \frac{W_2}{\sin 120^{\circ}}$$

$$\frac{46.18}{\sin 90^{\circ}} = \frac{T_{BC}}{\sin 150^{\circ}} = \frac{W_2}{\sin 120^{\circ}}$$

$$T_{BC} = \frac{46.18 \times \sin 150}{\sin 90^{\circ}} = 23 \text{ kN}$$

$$W_2 = \frac{46.18 \times \sin 120}{\sin 90^{\circ}} = 40 \text{ kN}$$
 (Ans)

(Ans)



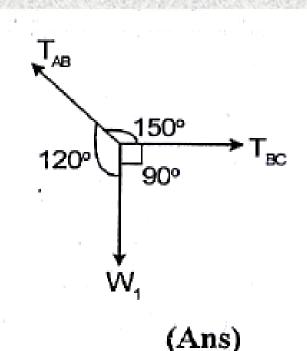


Now consider joint B, the various forces acting at 'B' is shown in fig. Applying Lami's Theorem,

$$\frac{T_{AB}}{\sin 90^{\circ}} = \frac{W_2}{\sin 150^{\circ}} = \frac{T_{BC}}{\sin 120^{\circ}}$$

$$T_{AB} = \frac{T_{BC} \times \sin 90^{\circ}}{\sin 120^{\circ}} = 26.56 \text{ kN}$$

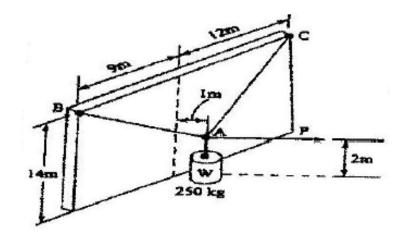
$$W_1 = \frac{T_{BC} \times \sin 150}{\sin 120^\circ} = 13.28 \text{ kN}$$







A horizontal force P normal to the wall holds the cylinder in the position shown in figure below. Determine the magnitude of P and the tension in each cable. (AU DEC'12)



Given:

Weight, $W = 250kg = 250 \times 9.81 = 2452.5N$

To find:

- i) Magnitude of P = ?
- ii) Tension in each cable





Solution:

Let T_{AB} and T_{AC} be the forces along the cables AB and AC respectively. The co-ordinates of various points are A(1, 2, 0), B(0, 14, 9), C(0, 14, -12).

Tension in Cable AB:

Position Vector
$$\overrightarrow{AB} = \overrightarrow{B} - \overrightarrow{A}$$

$$= (0i + 14j + 9k) - (i + 2j + 0k)$$

$$\overrightarrow{AB} = -i + 12j + 9k$$
Magnitude of $\overrightarrow{AB} = |\overrightarrow{AB}|$

$$= AB = \sqrt{(-1)^2 + 12^2 + 9^2}$$

$$= 15.03$$





Unit vector along AB,

$$n_{AB} = \frac{\overline{AB}}{|\overline{AB}|}$$

$$= \frac{-i + 12j + 9k}{15.03}$$

$$= -0.0665 i + 0.7984 j + 0.5988 k$$

Tension along AB,
$$\overline{T_{AB}} = T_{AB}$$
 . n_{AB}

$$= T_{AB} (-0.665 i + 0.7984j + 0.5988k)$$

$$= -0.665T_{AB}i + 0.7984T_{AB}j + 0.5988T_{AB}k$$

Tension in Cable AC:

Position Vector
$$\overrightarrow{AC}$$
 = $\overrightarrow{C} - \overrightarrow{A}$
= $(0i + 14j - 12k) - (i + 2j + 0k)$
 \overrightarrow{AC} = $-i + 12j - 12k$





Magnitude of
$$\overrightarrow{AC} = |\overrightarrow{AC}|$$

$$= AC = \sqrt{(-1)^2 + 12^2 + (-12)^2}$$

$$AC = 17$$

Unit vector along AC,

$$n_{AC} = \frac{\overline{AC}}{|\overline{AC}|}$$

$$= \frac{-i+12j-12k}{17}$$

$$= -0.0588 i + 0.7058 j - 0.7058k$$

Tension along AC,
$$\overline{T_{AC}} = T_{AC} \cdot n_{AC}$$

$$T_{AC} = T_{AC} (-0.0588 i + 0.7058 j - 0.7058k)$$

=
$$-0.0588T_{AC}i + 0.7058T_{AC}j - 0.7058T_{AC}k$$

Force through weight,

$$\overrightarrow{\mathbf{W}} = \mathbf{W} (-\mathbf{j}) = -2452.5\mathbf{j}$$





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Force through P,
                    \vec{\mathbf{p}} = \mathbf{P} \cdot \mathbf{i}
using equations of equilibrium 🕝
\sum F_x = 0
      -0.665 T_{AB} - 0.0588 T_{AC} + P = 0 ......(1)
\sum F_y = 0
      0.7984 T_{AB} + 0.7058 T_{AC} - 2452.5 = 0 \dots (2)
 \sum F_z = 0
       0.5988 T_{AB} - 0.7058 T_{AC} = 0 ......(3)
 Solving equations (1), (2), & (3) we get
                  T_{AB} = 1755.8 \text{ N}
                   T_{AC} = 1489.1 N
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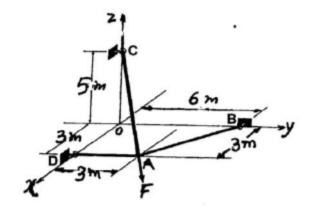
= 204.3 N

(Ans)





Figure below shows three cables AB, AC, AD that are used to support the end of a sign which exerts a force of $\vec{F} = \{250i + 450j - 450k\}N$ at A. Determine the force develop in each cable. (AU DEC'11)



Solution:

Let F_{AB} , F_{AC} and F_{AD} be the forces acting along cables A!3, AC and AD respectively.

From the geometry of figure, the co-ordinates of various points are

A (3,0,3), B (6,0,0), C (0,5,0), D (0,0,3)





Force Acting along AB

Position vector of AB is,
$$\overline{AB} = \overline{B} - \overline{A}$$

= $(6i-0i+0k) - (3i+0i+3k)$
 $\overline{AB} = 3i-3k$

Magnitude of
$$\vec{AB} = |\vec{AB}| = AB = \sqrt{3^2 + (-3)^2} = 4.24$$

Unit vector along AB is
$$n_{AB} = \frac{\overline{AB}}{|\overline{AB}|}$$

$$= \frac{3i - 3k}{4.24}$$

$$n_{AB} = (0.7075i - 0.7075k)$$

Force in wire AB,
$$\overrightarrow{F_{AB}} = F_{AB} \cdot n_{AB}$$

$$= F_{AB} (0.7075i - 0.7075k)$$

$$= 0.7075 \text{ F}_{AB} \text{ i} - 0.07075 \text{ F}_{AB} \text{k}$$





Force acting along AC

Position vector of AC is
$$\overrightarrow{AC} = \overrightarrow{C} - \overrightarrow{A}$$

= $(0i+5j+0k)-(3i+0j+3k)$
 $\overrightarrow{AC} = -3i +5j-3k$

Magnitude of
$$\overrightarrow{AC} = |\overrightarrow{AC}| = AC = \sqrt{(-3)^2 + 5^2 + (-3)^2} = 6.55$$

Unit vector acting along AC is $n_{AC} = \frac{\overrightarrow{AC}}{|\overrightarrow{AC}|}$

$$= \frac{3i + 5j - 3k}{6.55}$$





$$n_{AC} = -0.458i + 0.763j - 0.458k.$$

Force in wire AC,
$$\overline{F_{AC}} = F_{AC}.n_{AC}$$

= $F_{AC}(-0.458i+0.763j-0.458k)$
 $\overline{F_{AC}} = -0.458 F_{AC}i+00763 F_{AC}j-0.458 F_{AC}k$

Force acting along AD

Position vector of AD is
$$\overrightarrow{AD} = \overrightarrow{D} - \overrightarrow{A}$$

$$= (0i+0j+3k) - ()3i+0j+3k)$$

$$\overrightarrow{AD} = -3i$$

Magnitude of
$$\overrightarrow{AD} = |\overrightarrow{AD}| = AD = \sqrt{(-3)^2} = 3$$

Unit vector acting along AD is
$$n_{AD} = \frac{\overline{AD}}{|\overline{AD}|} = \frac{-3^{\circ}}{3} = -1^{\circ}$$

Force in wire AD,
$$\overrightarrow{F_{AD}} = F_{AD}.n_{AD}$$

$$\overline{F_{AC}} = -F_{AD}i$$
It is given that force at A is

$$\vec{F} = 250i + 450j - 150k$$





It is given that force at A is

$$\vec{F} = 250i + 450j - 150k$$

Applying equations of equilibrium,

$$\Sigma F_X = 0.7075 F_{AB} - 0.458 F_{AC} - F_{AD} + 250 = 0 \dots (1)$$

$$\Sigma F_{\mathcal{Y}} = 0.763 F_{AC} + 450 = 0 \dots (2)$$

$$\Sigma F_z = -0.7075 F_{AB} - 0.458 F_{AC} - 150 = 0 \dots (3)$$

Solving equations (1), (2) & (3) we get,

$$\mathbf{F}_{\mathbf{AC}} = 589.77\mathbf{N} \tag{Ans}$$

$$\mathbf{F}_{\mathbf{A}\mathbf{B}} = 593.8\mathbf{N} \tag{Ans}$$

$$\mathbf{F_{AD}} = 400\mathbf{N} \tag{Ans}$$





A force (10i+20j-5k)N acts at a point P (4,3,2) m. Determine the moment of this force about the point Q(2,3,4) m in the vector form, Also find the magnitude of the moment andits angles with respect to x,y,z axes. (AU Dec'10,JUN'12)

Given :

Force, $\vec{F} = (10i + 20j - 5k)N$ P (4,3,2) m

Q (2,3,4) m

To Find:

Moment, $\overline{M_0} = ?$

Magnitude of moment, $|\overline{M_Q}| = M = ?$ $\theta_x = ?$, $\theta_y = ?$ $\theta_z = ?$

Solution:

Position vector, $\overrightarrow{QP} = \overrightarrow{P} - \overrightarrow{Q}$

= (4i + 3j + 2k) - (2i + 3j + 4k) $\overrightarrow{OP} = 2i + 0j - 2k$

Moment about point Q, $\overline{M}_Q = \overline{QP} \times \overline{F} = \begin{vmatrix} i & j & k \\ 2 & 0 & 2 \\ 10 & 20 & -5 \end{vmatrix}$

$$= i (0 + 40) - j (-10 + 20) + k (40 - 0)$$

 $\overline{M}_{Q} = (40i - 10j + 40k) \text{ N-m.}$ (Ans)

Here, $M_x = 40 \text{N-m}$; $M_y = -10 \text{ N-m}$, $M_z = 40 \text{ N-m}$





Magnitude of moment =
$$|\overline{M}_{Q}| = M = \sqrt{(40)^2 + (-10)^2 + (40)^2}$$

 $M = 57.44 \text{ N-m}$
Angle made by \overline{MO} with x-axis
$$\cos \theta x = \frac{Mx}{M}$$

$$\theta x = \cos^{-1}\left(\frac{40}{57.44}\right)$$

$$\theta x = 45.86^{\circ}$$
Similarly,
$$\cos \theta y = \frac{My}{M}$$

$$\theta y = \cos^{-1}\left(\frac{-10}{57.44}\right)$$

 $\theta y = 100.02^{\circ}$

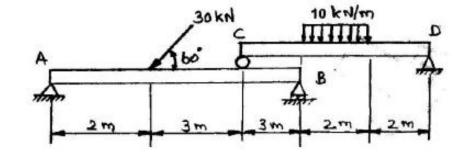
$$\theta z = \cos^{-1}\left(\frac{Mz}{M}\right) = \cos^{-1}\left(\frac{40}{57.44}\right)$$

$$\theta z = 45.86^{\circ}$$
 (Ans)





Two beams AB and CD are shown in figure. A and D are hinged supports. B and C are roller supports.



- Sketch the free body diagram of the beam AB and determine the reactions at the supports A & B.
- (ii) Sketch the free body diagram of beam AB and determine the reactions at the supports C and D. (AU Dec'10,DEC'12)

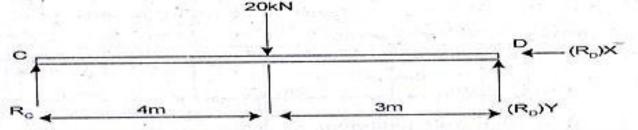




Solution :

Let R_A, R_B, R_C and R_D be the reactions at supports A, B, C and D respectively.

Consider the beam CD. The free body diagram of beam CD is shown below.



The uniformly distributed load of 10 kN/m for a length of 2 m is assumed as equivalent point load of (10×2=20kN) and acting at a distance 4 m form C.

Using equations of equilibrium,

$$\Sigma F_{x} = 0$$
 (Ans)
 $\sum F_{y} = 0$ (Ans)
 $R_{C} - 20 + (R_{D})_{y} = 0$ (1)
Taking moments about C,

$$\sum M_C = 0$$

-20×4+(R_D)_v ×7 = 0

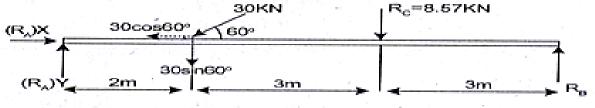




$$-80+7(R_D)_y = 0$$

 $(R_D)_y = 11.43 \text{ kN}$ (Ans)
Put $(R_D)_y = 11.43 \text{ in eqn. (1), we get}$
 $R_C + 11.43 = 20$
 $R_C = 8.57 \text{ kN}$ (Ans)

Now consider the beam AB. The free body diagram of beam AB is shown below



Using equations of equilibrium,

-51.96-42.85+8 R_B = 0

 $\mathbf{R}_{\mathbf{R}}$

$$\Sigma F_{X} = 0$$

$$(R_{A})_{x} - 30 \cos 60 = 0$$

$$(R_{A})_{x} = 15 \text{ kN}$$

$$\Sigma F_{y} = 0$$

$$(R_{A})_{y} - 30 \sin 60 - 8.57 + R_{B} = 0$$

$$(R_{A})_{y} + R_{B} = 34.55 \qquad (2)$$
Taking moments about A,
$$\Sigma M_{A} = 0$$

$$-30 \sin 60 \times 2 - 8.57 \times 5 + R_{B} \times 8 = 0$$

- 11.85 kN

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(Ans)

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Put
$$R_B = 11.85$$
 in eqn. (2), we get

$$(R_A)_y + 11.85 = 34.55$$

$$(R_A)_v = 22.7 \text{ kN}$$

$$R_A = \sqrt{(R_A)_x^2 + (R_A)_y^2}$$

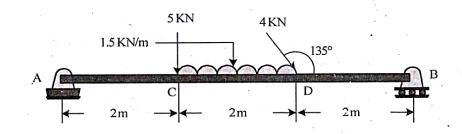
$$= \sqrt{(15)^2 + (22.7)^2}$$

$$R_{A} = 27.2 \text{ kN} \qquad (Ans)$$





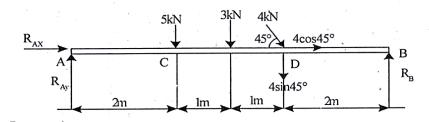
A simply supported beam AB of 6m span is loaded as shown A is a hinged support; B is a roller support. Determine the reactions at A and B.(AU MAY'11)



Solution:

Let R_{AX} and R_{AY} be horizontal and verticle component of reaction R_A at hinged support A.

Let R_B be the vertical component at B due to roller support.



The uniformly distributed load of 1.51kN/m for a length of 2m is assumed as equivalent point load of $(1.5 \times 2 = 3 \text{kN})$ and acting at a distance $\frac{2}{2} = 1 \text{m}$ from 'C'.





Using equations of equilibrium

Taking moments about A,

$$\Sigma M_A = 0$$

$$-5 \times 2 - 3 \times 3 - 4 \sin 45^{\circ} \times 4 + R_{B} \times 6 = 0$$
.

$$-10-9-11.31+6R_{\rm B} = 0$$

$$R_{\rm R} = 5.05 kN$$

$$\Sigma \mathbf{F}_{\mathbf{x}} = \mathbf{0}$$

$$R_{Ax}+4 \cos 45^{\circ} = 0$$

$$R_{Ax} = -3.98 \text{ kN}$$

$$\Sigma F_y = 0$$

$$R_{Ay} - 5 - 3 - 4\sin 45^{\circ} + 4R_{B} = 0$$

$$R_{Ay} = 5.778kN$$

Magnitude of Reaction, RA
$$= \sqrt{R_{AX}^2 + R_{Ay}^2}$$
$$= \sqrt{(3.98)^2 + (5.77)^2}$$

$$R_A = 7 \text{ kN}$$
 (Ans)

(Ans)

Direction of R_A is

$$\theta = \tan^{-1} \left(\frac{R_{Ay}}{R_{Ax}} \right)$$

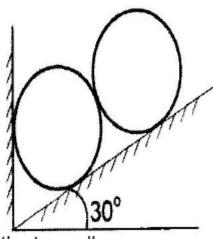
$$\theta = \tan^{-1}\left(\frac{5.778}{3.98}\right)$$

$$\theta = 55.4 \tag{Ans}$$





Two identical rollers, each of weight 500N, are supported by an inclined plane making an angle of 30° to the horizontal and a vertical wall as shown in the figure.(AU Jun'10,DEC'12)



- (i) Sketch the free body diagrams of the two rollers.
- (ii) Assuming smooth surfaces, find the reactions at the support points.





Given:

Weight of two identical rollers, W = 5.00 N

To Find:

Reactions at supports.

Solution:

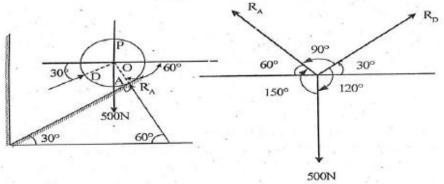
Let RA, RB and RC be the reactions at supports.

Considering the roller P

The freebody diagram is shown in the figure.

Free body diagram

Force diagram



Applying Lami's theorem

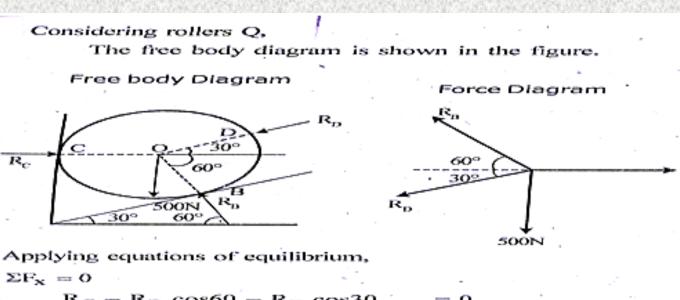
$$\frac{R_{A}}{\sin 120} = \frac{R_{B}}{\sin 150} = \frac{500}{\sin 90}$$

$$\frac{R_A}{\sin 120} = \frac{500}{\sin 90} \implies R_A = 433 \text{ N}$$

$$\frac{R_D}{\sin 150} = \frac{500}{\sin 90} \implies R_D = 250 \text{ N}$$







$$R_{\rm C} - R_{\rm B} \cos 60 - R_{\rm D} \cos 30 = 0$$

$$R_{\rm C} = 0.5 R_{\rm B} = 250 \times 0.866 = 0$$

 $R_{\rm C} = 0.5 R_{\rm B} = 216.5$

Resolving forces vertically,

$$R_{\rm B} \sin 60 - R_{\rm D} \sin 30 - 500 = 0$$

 $0.866 R_{\rm B} - 250 \times 0.5 - 500 = 0$

Put
$$R_B = 721.7$$
 in equation (1)
 $R_C = 0.5$ (721.7)

- 721.7 N

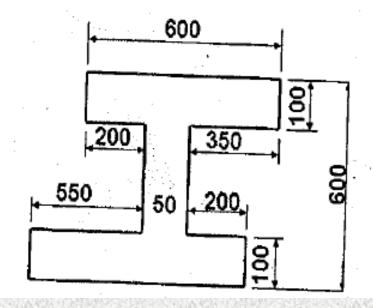
....CD

 $\Sigma F_{\vee} = 0$.





For the section shown in figure below, locate the horizontal and vertical centroidal Axis (AU JUN'12)

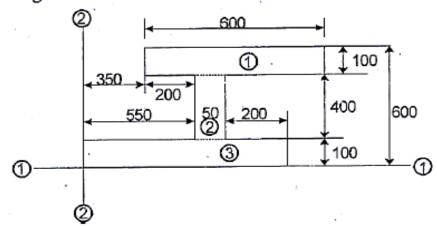






Solution:

The given section is split into three rectangles as shown in the figure. Let 1-1 and 2-2 be the reference axis



Component	Area a (mm²)	Centroidal Distance from 2-2 axis 'x' (mm)	Centroidal Distance from 1-1 axis 'y' (mm)	ax (mm²)	ay (mm²)
Rectangle (1)	100×600 = 60000	$\frac{600}{2} + 350 = 650$	$\frac{100}{2} + 500 = 550$	39×10 ⁶	33×10 ⁶
Rectangle (2)	400×50 = 20000	$\frac{50}{2} + 550 = 575$	$\frac{400}{2} + 100 = 300$	11.5×10°	6×10 ⁶
Rectangle (3)	800×100 = 80000	$\frac{800}{2} = 400$	$\frac{100}{2} = 50$		
	$\Sigma a = 160 \times 10^3$				$\Sigma ay = 43 \times 10^6$





Centroidal distance from reference axis, 2-2,

$$\frac{\overline{x}}{=} \frac{\sum ax}{\sum a}$$

$$= \frac{82.5 \times 10^6}{160 \times 10^3}$$

$$= 515.625 \text{ mm}$$

Centroidal distance from reference axis, 2-2,

$$\overline{y} = \frac{\sum ay}{\sum a}$$

$$= \frac{43 \times 10^6}{160 \times 10^3}$$

$$\overline{y} = 268.75 \text{ mm}$$

(Ans)

(Ans)





Calculate the centroidal polar moment of inertia of a rectangular section with breadth of 100 mm and height 200 mm. (AU DEC'10,JUN'12)

Given:

For Rectangular section

Breadth, b = 100mm

Height, h = 200 mm

To Find:

Polar moment of Inertia, J = ?

Solution:

M.I of rectangle about x-x axis

$$I_{xx} = \frac{bh^3}{12} = \frac{100 \times 200^3}{12} = 66.67 \times 10^6 \text{mm}^4$$

M.I of rectangle about y-y axis

$$I_{yy} = \frac{hb^3}{12} = \frac{200 \times 100^3}{12} = 16.67 \times 10^6 \text{mm}^4$$

Polar moment of Inertia is,

$$J = I_{xx} + I_{yy}$$

= 66.67×10⁶+16.67×10⁶

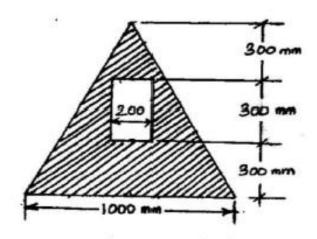
$$J = 83.34 \times 10^6 \text{mm}^4$$

(Ans)





Find the moment of inertia of the shaded area shown in figure about the vertical and horizontal centroidal axes. The width of the hole is 200 mm. (AU DEC'12, JUN'10)



As the given section is symmetrical about y-y axis its centroid will lie on this axis. Let 1-1 be the reference axis





As the given section is symmetrical about y-y axis its centroid will lie on this axis. Let 1-1 be the reference axis

Component	Area a (cm²)	Centroidal Distance from 1-1 axis 'y' (cm)	а <i>у</i> (cm²)	I _{self} about the axis x-x (cm ⁴)
Triangle	$a_1 = \frac{1}{2} bh$	$y_1 = \frac{h}{3} = \frac{90}{3}$	135000	$l_{selft} = \frac{bh^3}{36}$
,	$=\frac{1}{2} \times 90 \times 100$ = 4500	= 30		$=\frac{100 \times 90^3}{36}$ $= 2025000$
Rectangle	a ₂ 20×30 = -600	$y_2 = \frac{30}{2} + 30$ = 4.5	-27000	$ 1_{\text{self2}} = \frac{\text{bd}^3}{12} \\ = \frac{20 \times 30^3}{12} \\ = 45000 $
	Σa= 39000		Σay = 108000	

Distance of centroidal axis x-x from 1-1 axis

$$\overline{y} = \frac{\sum ay}{\sum a}$$

$$= \frac{108000}{3900}$$

$$\overline{y} = 27.7 \text{ cm} \quad \text{(Ans)}$$





M.I of entire section about x-x axis

 I_{x-x} = M.I of Triangle about x-x axis–M.I of Rectangle about x-x axis $[I_{selfl} + a_1(y_1 - \bar{y})^2] + [I_{self2} + a_2(y_2 - \bar{y})^2]$

= $[2025000+4500(30-27.7)^2]-[45000 + 600(45-27.7)^2]$

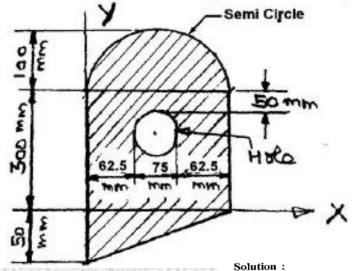
= 2048805 - 224574

 $I_{y=x} = 1824231 \text{ cm}^4$ (Ans)

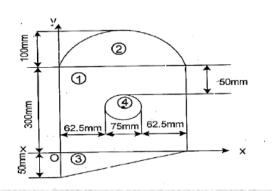




Locate the centroid of the plane area shown in figure below. (AU DEC'11, DEC'12)



The given section is split up into four sections as shown in figure. Let OX and OY be the reference axis.







Component	Area 'a' (mm²)	Centroidal Distance from O-Y axis 'x' (mm)	Centroidal Distance from O-X axis 'y' (mm)	ax (mm²)	ay (mm²)
Rectangle	300×200 = 60×10 ³	$\frac{200}{2} = 100$	$\frac{300}{2} = 150$	6×10 ⁶	9×10 ⁶
Semi-circle	$ \frac{\frac{\pi r^2}{2}}{\frac{\pi \times 100^2}{2}} = 15.7 \times 10^3 $	$\frac{200}{2} = 100$	$\frac{\frac{4r}{3\pi} + 300}{\frac{4 \times 100}{3\pi} + 300}$ = 342.44	1.57 ×10 ⁶	5.37 ×10 ⁶
Triangle	$\frac{1}{2} \times 200 \times 50$ =5000	$\frac{200}{3} = 66.67$	$\frac{50}{3} = -16.67$	0.33 ×10 ⁶	−83.35 ×10 ³
Circle (-)	$-\pi r^{2}$ = $-\pi \times 100^{2}$ = -31.41×10^{3}	200 2 =100	$300 - \left(50 + \frac{75}{2}\right)$ =212.5	−3.14 ×10 ⁶	−6.67 ×10 ⁶
	Σa=49.29×10 ³		_	Σax= 4.76×10 ⁶	Σay = 7.61×10 ⁶





Centrodial distance from O-Y axis,

$$\overline{x} = \frac{\sum ax}{\sum a} = \frac{4.76 \times 10^6}{49.29 \times 10^3} = 96.57 \text{mm}$$

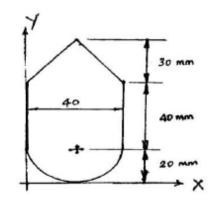
Centrodial distance from Q-X axis,

$$\overline{y} = \frac{\sum ay}{\sum a} = \frac{7.61 \times 10^6}{49.29 \times 10^3} = 154.39 \text{mm}$$





Figure shows a composite area. (AU DEC'11, JUN' 10)



Find the moments of inertia (second moments of area) about both the centroidal axes.

Solution: The given section is split into three sections as shown in the figure let Ox and Oy be the reference axis.





Centrodial distance from O-y axis,

$$\overline{x} = \frac{\sum ax}{\sum a} = \frac{56.56 \times 10^3}{2828.32} = 20$$
mm

Centrodial distance from O-Y axis,

$$\overline{y} = \frac{\sum ay}{\sum a} = \frac{77.23 \times 10^3}{2828.32} = 27.3$$
mm

To find moment of Inertia about X-X axis:

=
$$[Iself_1 + a_1(y_1 - \overline{y})^2] + [Iself_2 + a_2(y_2 - \overline{y})^2]$$

$$= [0.2136 \times 10^{6} + 1600(40 - 27.3)^{2}] +$$

+ $[Iself_2 + a_2(y_3 - \overline{y})^2]$

$$[30\times10^3+600(10-27.3)^2]$$
 +

$$[17.6 \times 10^{3} + 628.32(11.51 - 27.3)^{2}]$$
= 471.664 \times 10^{3} + 209.574 \times 10^{3} + 174.255 \times 10^{3}

$$I_{yy} = 855.49 \times 10^3 \text{ mm}^4$$
 (Ans)

To find moment of Inertia about Y-Y axis





Component	Area 'a' (mm²)	Centroidal Distance from O-Y 'x' (mm)	Centroidal Distance from O-X 'y' (mm)	áv (mm³)	ay (mm³)	Itself about x-x axis mm ⁴	Itself about y-y axis mm ⁴
(1) Rectangle	40×40 = 1600	$\frac{40}{2} = 20$	$\frac{40}{2} = +20$ $= 40$. 32×10 ³	64×10 ³	$ \frac{bd^{3}}{12} = \frac{40 \times 40^{3}}{12} = 0.213 \times 10^{6} $	$ \frac{bd^{3}}{12} = \frac{40 \times 40^{3}}{12} = 0.213 \times 10^{6} $
(2) Triangle	$\frac{1}{2} \times 40 \times 30$ $= 600$	$\frac{40}{2} = 20$	$\frac{30}{3} = 10$	12×10 ³	6×10 ³	$\frac{hb^{3}}{36} = \frac{40 \times 30^{3}}{36} = 30 \times 10^{3}$	$\frac{hb^3}{36} = \frac{30 \times 40^3}{48} = 40 \times 10^3$
(3) Semi circ's	$\frac{\pi r^2}{\frac{2}{2}} = \frac{\pi \times 20^2}{2} = 628.32$	$\frac{40}{2} = 20$	$20 - \frac{4r}{3\pi} = 20 - \frac{4 \times 20}{3\pi} = 11.51$	12.56×10 ³	7.23×10 ³	$0.11r^4$ = 0.11×20 ⁴ = 17.6×10 ³	$\frac{\pi D^4}{128} = \frac{\pi (40)^4}{128} = 62.83 \times 10^3$
	Σa = 2828.32		1	$\Sigma ax = 56.56 \times 10^3$	Σay = 77.23×10 ³		

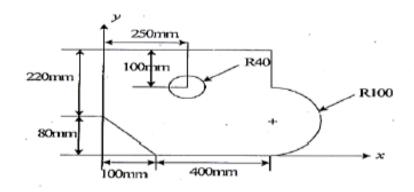
=
$$[Iself_1 + a_1(x_1 - \overline{x})^2] + [Iself_2 + a_2(x_2 - \overline{x})^2] + [Iself_3 + a_3(x_3 - \overline{x})^2]$$

= $[0.213 \times 10^6 + 1600(20 - 20)^2] + [40 \times 10^3 + 600(20 - 20)^2] + [62.83 \times 10^3 + 628.32(20 - 20)^2]$
= $0.213 \times 10^6 + 40 \times 10^3 + 62.83 \times 10^3$
 $I_{yy} = 315.83 \times 10^3 \text{mm}^4$ (Ans)



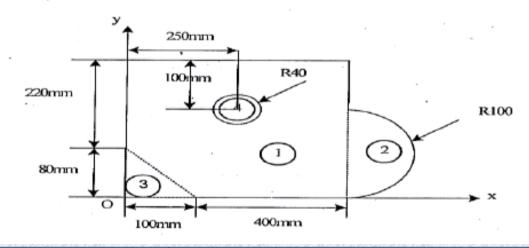


Locate the centroid of the plane area shown in figure below (AU MAY'11)



Solution:

The given section is split up into four sections as shown in figure. Let OX and OY be the reference axis.







Component	Area 'á' (mm²)	Centroidal Distance from O-Y axis 'x' (mm)	Centroidal Distance from O-X axis 'y' (mm)	ax (mm²)	ay (mm²)
Rectangle	300×500 = 150×10 ³	$\frac{500}{2} = 250$	$\frac{300}{2} = 150$	37.5×106	22.5×10 ⁶
Semi-circle	$\frac{\pi r^2}{\frac{2}{2}} = 15.7 \times 10^3$	$\frac{\frac{4r}{3\pi} + 500}{\frac{4 \times 100}{3\pi} + 500}$ = 542.44	100	8.5 ×10 ⁶	1.57 ×10 ⁶
Triangle	$\frac{1}{2} \times 100 \times 80$ $= -4 \times 10^{3}$	$\frac{100}{3} = 33.34$	$\frac{80}{3}$ =26.67	-0.133 ×10 ⁶	−0.106 ×10 ³
Circle (-)	$-\pi r^{2}$ $= -\pi (40)^{2}$ $= -5.02 \times 10^{3}$	250	300–100 =200	-1.255 ×10 ⁶	×10 ⁶
	Σa=156.68×10	3		Σax= 82.5×10 ⁶	Σay =. 22.96×10

Distance of centroid from OY axis, $\overline{x} = \frac{\sum ax}{\sum a}$



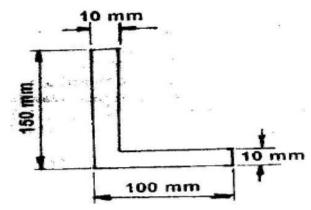


$$= \frac{44.61 \times 10^6}{156.68 \times 10^3} = 284.72 \text{mm}$$
 (Ans)

Distance of centroid from OX axis,
$$\overline{y} = \frac{\sum ay}{\sum a}$$

$$= \frac{22.96 \times 10^6}{156.68 \times 10^3} = 146.54 \text{mm}$$
 (Ans)

An area in the form of L section is shown in figure below (AU MAY'11, DEC'12)



Find the moments of Inertia I_{xx} , I_{yy} , and I_{xy} about its centroidal axes. Also determine the principal moments of inertia.

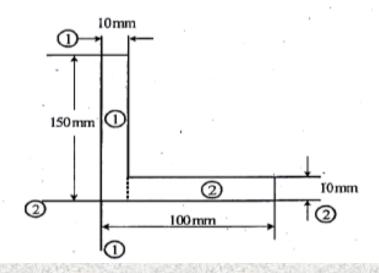




Find the moments of Inertia I_{xx} , I_{yy} , and I_{xy} about its centroidal axes. Also determine the principal moments of inertia.

Solution:

The given section is split up into two rectangles as shown in figure. Let 1-1 and 2-2 be the reference axis. The calculations are shown in the table







Component	Area 'a' (mm²)	Centroidal Distance from O-Y axis 'x' (mm)	Centroidal Distance from O-X axis 'y' (mm)	ax (mm²)	ay (mm²)	L _{self} alxout x-x axis mm ⁴	I _{self} about y-y axis mm ⁴
Rectangle	150×10 =1500	$\frac{10}{2} = 5$	$\frac{150}{2} = 75$	7500	112.5 ×10 ³	$ \frac{bd^{3}}{12} \\ \frac{10 \times 150^{3}}{12} \\ = 2.81 \times 10^{6} $	$ \frac{\frac{db^3}{12}}{\frac{150 \times 10^3}{12}} \\ = 12.5 \times 10^3 $
Rectangle (2)	90×10 =900	=55	$\frac{10}{2} = 5$	49.5 ×10 ³	4500	$ \frac{bd^{3}}{12} $ $ \frac{90 \times 10^{3}}{.12} $ $ =7.5 \times 10^{3} $	$ \frac{db^3}{12} \\ 10 \times 90^3 \\ 12 \\ = 60.75 \times 10^3 $
	Σa == 2400			Σax= 57×10 ³	Σay = 117×10 ³	7.5% 10	

1. To find centroidal distance:

Distance of centroidal axis y-y from 1-1 axis,

$$\overline{x} = \frac{\Sigma ax}{\Sigma a} = \frac{57 \times 10^3}{2400} = 23.75 \text{mm}$$





1. To find centroidal distance:

Distance of centroidal axis y-y from 1-1 axis,

$$\overline{x} = \frac{\Sigma ax}{\Sigma a} = \frac{57 \times 10^3}{2400} = 23.75 \text{mm}$$

Distance of centroidal axis x-x from 2-2 axis,

$$\overline{y} = \frac{\Sigma a y}{\Sigma a} = \frac{117 \times 10^3}{2400} = 48.75 \text{ mm}$$

2.To find moment of inertia about centroidal x-x axis (I_{xx})

M.I. of given section about centroidal x-x axis is

 $I_{xx} = M.I.$ of rectangle (1) about x-x axis + M.I of rectangle (2) about x-x axis.

=
$$[I_{\text{self1}} + a_1(y_1 - \overline{y})^2] + [I_{\text{self2}} + a_2(y_2 - \overline{y})^2]$$





$$= [2.81 \times 10^{6} + 1500(75 - 48.75)^{2}] + [7.5 \times 10^{3} + 900(5 - 48.75)^{2}]$$

$$= 3.84 \times 10^{6} + 1.73 \times 10^{6}$$

$$I_{xx} = 5.57 \times 10^{6} \text{mm}^{4}$$
(Ans)

3.To find moment of inertial about centroidal y-y axis (I_{vv})

 $I_{yy} = M.I.$ of rectangle (1) about y-y axis + M.I of rectangle (2) about y-y axis.

=
$$[I_{\text{self1}} + a_1(x_1 - \overline{x})^2] + [I_{\text{self2}} + a_2(x_2 - \overline{x})^2]$$

= $[12.5 \times 10^3 + 1500(5 - 23.75)^2][60.7.5 \times 10^3 + 900(55 - 23.75)^2]$
= $0.539 \times 10^6 + 1.486 \times 10^6$

$$I_{yy} = 2.025 \times 10^6 \text{mm}^4$$
 (Ans)





4) To find product of inertia (I_{XY}) :-

Product of inertia of given section is

$$I_{XY} = [I_{x_1'y_1'} + a_1x_1y_1] + [I_{x_2'y_2'} + a_2x_2y_2]$$

where,

 x_1', x_2', y_1' and y_2' are the axes of symmetry.

$$I_{x_1'y_1'} = 0$$
, $I_{x_2'y_2'} = 0$,

$$I_{xy} = [0 + 1500 \times 5 \times 75] + [0 + 900 \times 55 \times 5]$$
$$= 0.5625 \times 10^6 + 0.2475 \times 10^6$$

$$I_{xy} = 8.1 \times 10^5 \text{ mm}^4$$
 (Ans)





5) To taid principal moment of inertia:-

The principal moments of inertia is given by

$$I_{\text{max, min}} = \frac{I_{xx} + I_{yy}}{2} \pm \sqrt{\left(\frac{I_{xx} - I_{yy}}{2}\right)^2 + I_{xy}^2}$$

$$= \frac{5.57 \times 10^6 + 2.025 \times 10^6}{2} \pm \sqrt{\left(\frac{5.57 \times 10^6 - 2.025 \times 10^6}{2}\right) + (8.1 \times 10^5)^2}$$

$$= 3.797 \times 10^6 + 1.948 \times 10^6$$

$$= 3.797 \times 10^6 \pm 1.948 \times 10^6$$

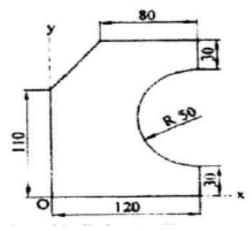
$$I_{\text{max}} = 3.797 \times 10^6 + 1.948 \times 10^6 = 5.745 \times 10^6 \text{ mm}^4$$

$$I_{min} = 3.797 \times 10^6 - 1.948 \times 10^6 = 1.849 \times 10^6 \text{ mm}^4 \text{ (Ans)}$$

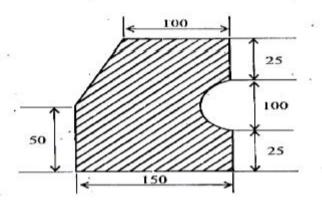




Locate the centroid of the area shown in figure below. The dimensions are in mm. (AU JUN'10,DEC 11)

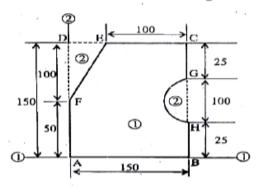


Solution :



The given section is not symmetrical about any axis Hence, we have to find the values of \bar{x} and \bar{y} . Split up the area into square, triangle and semi-circle as in fig. The area is obtained by adding square and subtracting both the triangle and semi-circle.

The calculations are shown in the following table.







2.0	Componen	Area 'a' (mm²)	Distance o centroid from 2-2 'x' (mm)	f Distance of centroid from 1-1 'y' (mm)	ax (mm³)	ay (mm³)
i.	Square ABCD	150x150 =22.5x10 ³	150 2 = 75	150 2 = 75	1687.5×103	1687.5×10 ³
2.	Semi Circle	2 gtr ²	$150 - \frac{4r}{3\pi}$	$\frac{100}{2}$ + 25		
		$= \frac{-\pi \times 50^2}{2}$ $= -3.927 \times 10^3$	$=150 - \frac{4 \times 50}{3\pi}$ = 128.78	- 75	-505.71×103	-294.525×103
з.	Triangle	- 1/2 bh	b/3 .	150 - h	-41.662×10 ³	-291.665×10°
	ADE	$= \frac{1}{2} \times 50 \times 100$ $= -2.5 \times 10^{3}$	$=\frac{50}{3}$ = 16.66	$150 - \frac{100}{3}$ = 116.66		
		Σa - 16.073 x 10 ³			Σax = 1140.12×10'	Σay = . 101.31x10 ³

Centroid distance from 2-2 axis,

$$\frac{\sum ax}{\sum a}$$

$$= \frac{1140.12 \times 10^3}{16.073 \times 10^3} \qquad \overline{x} = 70.93 \text{ mm (Ans)}$$

Centroidal distance from 1-1 axis,

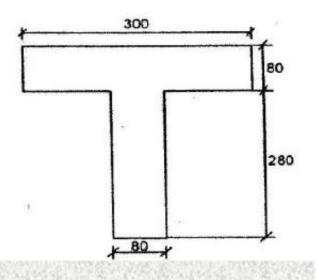
$$\overline{y} = \frac{\sum ay}{\sum a}$$

$$= \frac{1101.31 \times 10^3}{16.073 \times 10^3} \qquad \overline{y} = 68.52 \text{ mm (Ans)}$$



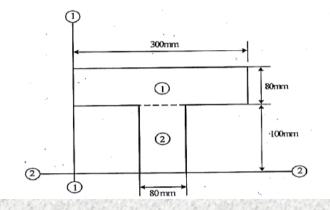


Find the polar moment of inertia of a T section shown in Fig 5 about an axis passing through its centroid. Also find the radius of gyration with respect to the polar axis. (Dimensions in mm) (AU JUN'09)



Solution:

The given section is split up into two rectangles as shown in the figure. Let 1-1 and 2-2 be the axis of reference. The calculations are shown in the table.







Component	Area 'a' (mm²)	Centroidal Distance from -1-1 axis 'x' (mm)	Centroidal Distance from 2-2 axis 'y' (mm)	ax (mnı³)	ay (mm³)	I _{self} about centroidal axis x-x mm ⁴	I _{self} about centroidal axis y-y mm ⁴
(I) Rectangle	300× 80 =24×10 ³	$\frac{300}{2}$ =150	$\frac{80}{2}$ +100	3.6×10 ⁶	3.36×10 ⁶	$\frac{bd^3}{12} = \frac{300 \times 80^3}{12}$ $= 12.8 \times 10^6$	12 12 =180×10 ⁶
(2) Rectangle	100×80 =8×10 ³	$\frac{300}{2}$ =150	$\frac{100}{2}$ =50	1.2×10 ⁶	0.4×10 ⁶	$\frac{bd^3}{12} = \frac{80 \times 100^3}{12}$ $= 6.67 \times 10^6$	$\frac{db^3}{12} = \frac{100 \times 80^3}{12}$ $= 4.26 \times 10^6$
,	$\Sigma a = 32 \times 10^3$			Σax = 4.8×10 ⁶	$\Sigma ay = 3.76 \times 10^6$		





To find centroid:

Distance of centroidal axis y-y from 1-1 axis

$$\overline{x} = \frac{\Sigma ax}{\Sigma a} = \frac{4.8 \times 10^6}{32 \times 10^3} = 150 \text{mm}$$

Distance of centroidal axis x-x from z-z axis

$$\overline{y} = \frac{\Sigma ay}{\Sigma a} = \frac{3.76 \times 10^6}{32 \times 10^3} = 117.5 \text{mm}$$

To find moment of inertia about centroidal x-x axis.

M.I. of the section about horizontal centroidal x-x axis is

$$= \left[I_{\text{self}1} + a_1 \left(\overline{y} - y_1\right)^2\right] + \left[I_{\text{self}_2} + a_2 \left(\overline{y} - y_2\right)^2\right]$$

=
$$[1.28 \times 10^6 + 24 \times 10^3 (117.5 - 140)^2] +$$

[$6.67 \times 10^6 + 8 \times 10^3 (117.5 - 50)^2$]

$$=24.95\times10^{6}+43.12\times10^{6}$$

$$I_{xx} = 68.07 \times 10^6 \text{ mm}^4$$





To find moment of inertia about centroidal y-y axis.

M.I. of section of about vertical centroidal y-y axis is

$$= \left[I_{\text{self}_1} + a_1\left(\overline{x} - x_1\right)^2\right] + \left[I_{\text{self}_2} + a_2\left(\overline{x} - x_2\right)^2\right]$$

=
$$[180 \times 10^6 + 24 \times 10^3 (150 - 150)] + [4.26 \times 10^6 + 8 \times 10^3 (150 - 150)^2]$$

$$=[180\times10^6+4.26\times10^6]$$

$$I_{yy} = 184.26 \times 10^6 \text{ mm}^4$$

To find polar moment of inertia:

Polar moment of inertia of given T-section is

$$\mathbf{J} = \mathbf{I}_{\mathbf{x}\mathbf{x}} + \mathbf{I}_{\mathbf{y}\mathbf{y}}$$

$$J = 68.07 \times 10^6 + 184.26 \times 10^6 = 252.38 \text{ mm}^4$$





To find radius of gyration with respect to polar axis:

Radius of gyration about x-x axis

$$K_{x-x} = \sqrt{\frac{I_{xx}}{\Sigma a}} = \sqrt{\frac{68.07 \times 10^6}{32 \times 10^3}} = 46.12 \text{ mm}$$

Radius of gyration about y-y axis.

$$K_{y-y} = \sqrt{\frac{I_{yy}}{\Sigma a}} = \sqrt{\frac{184.26 \times 10^6}{32 \times 10^3}} = 75.88 \text{ mm}$$

Radius of gyration about polar axis is

$$K_{p} = \sqrt{K_{xx}^{2} + K_{yy}^{2}}$$

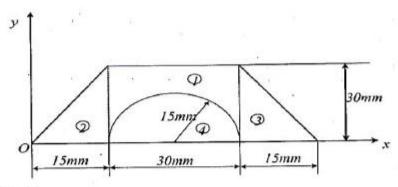
$$= \sqrt{(46.12)^{2} + (75.88)^{2}}$$

$$K_{p} = 88.79 \text{ mm} \quad \text{(Ans).}$$





Calculate the centroidal moment of inertia of the shaded area shown in figure below. (AU DEC'09,JUN'12)



Solution:

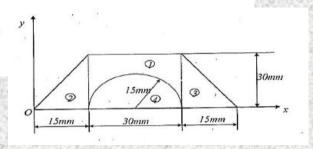
The given area is split up into various sections as shown in figure. Let OX and OY be the reference axis.

The calculations are shown in the table.

i. To find centroid:

Distance of Centroidal axis YY from O - Y is

$$\frac{1}{x} = \frac{\sum ax}{\sum a} = \frac{29.89 \times 10^3}{996.57} = 30 \text{mm}$$







Component	Area -a mm ²	Centroidal distance from 0-y axis 'x' (mm)	Centroidal distance from 0-x axis 'y' (mm)	ax mm ³	ay mm³	Iseif about x-x axis mm ⁴	 Iseif about y-y axis mm⁴
Rectangle (1)	30×30 = 900	$\frac{30}{2} + 5 = 30$	$\frac{30}{2} = 15$	27×10 ³	13.5×10 ³	$\frac{bd^3}{12} = \frac{30 \times 30^3}{12}$ $= 67.5 \times 10^3$	$\frac{db^3}{12} = \frac{30 \times 30^3}{12}$ $= 67.5 \times 10^3$
Triangle (2)	$\frac{1}{2} \times 15 \times 30$ $= .225$	$\frac{2}{3} \times 6 =$ $\frac{2}{3} \times 15 = 10$	$\frac{4}{3} = \frac{30}{3}$ = 10	2.25×10 ³	2.25×10 ³	$\frac{bh^3}{36} = \frac{15 \times 30^3}{36}$ $= 11,250$	$\frac{ab^3}{36} = \frac{30 \times 15^3}{36}$ $= 2812.5$
Triangle (3)	$\frac{1}{2} \times 15 \times 30$ $= 225$	$\frac{15}{3} + 30 + 15 = 50$	$\frac{30}{3} = 10$	11.25×10 ³	2.25×10 ³	$\frac{15 \times 30^3}{36} = 11,250$	$\frac{30 \times 15^3}{36} = 2812.5$
Semicircle (4) (–)	$\frac{51 \times 15^2}{2} = -(353.43)$	$\frac{30}{2} + 15$ = 30	$\frac{4\gamma}{3\pi} = \frac{4 \times 15}{3\pi}$ $= 6.366$	(10.602×10 ³)	(2.25×10 ³)	$0.11\gamma^{4}$ = 0.11×15 ⁴ = 5.56×10 ³	$\frac{\pi d^4}{128} = \frac{\pi \times 30^4}{128}$ $= 19.88 \times 10^3$
	$\Sigma_a = 996.57$			$\Sigma ax = 29.89 \times 10^3$	$\Sigma ay = 15.75 \times 10^3$		- 13.00^10





Distance of Centroidal axis XX from O - X is

$$\frac{1}{y} = \frac{\sum ay}{\sum a} = \frac{15.75 \times 10^3}{996.57} = 15.8 \text{ mm}$$

To find momentof Inertia about X - X axis:

M.I of entire section about X - X axis,

 $I_{X-X} = M.I$ of Rectangle (1) about x axis+ M.I of triangle (2) about x-x axis + M.I of triangle (3) about x-x axis - M.I of semi circle (4) about x-x axis

=
$$[Iself_1 + a_1(\overline{y} - y_1)^2] + [Iself_2 + a_2(\overline{y} - y_2)^2]$$

+ $[Iself_3 + a_3(\overline{y} - y_3)^2] + [Iself_4 + a_4(\overline{y} - y_4)^2]$

=
$$[67.5 \times 10^3 + 900(15.8 - 15)^2] + [11250 + 225(15.8 - 10)^2] + [11250 + 225(15.8 - 10)^2] - [5.56 \times 10^3 + 353.43(15.8 - 6.36)^2]$$

$$= 68076 + 18819 + 18819 - 37055.42$$

$$I_{XX} = 68.65 \times 10^3 \text{ mm}^4$$
 (Ans)





Two trains A and B leave the same station on parallel lines. A starts with a uniform acceleration of 0.15m/s² and attains the speed of 24 km/hour after which its speed remains constant. B leaves 40 seconds later with uniform acceleration of 0.30 m/s² to attain a maximum of 48 km/hour, its speed also becomes constant thereafter. When will B overtake A.(AU,Dec'11,JUN 12)

Solution:

```
Consider the motion of Train A:
```

Initial velocity, u = 0

Final velocity, V = 24 km/hr

$$= \frac{24 \times 1000}{3600} = 6.67 \text{m/s}^2$$

Acceleration, $a = 0.15 \text{m/s}^2$

T= time taken when the train B will overtake the train A from its start.

 t_A = time taken by train A to attain a speed of 6.67 m/s²

$$\vee = \mathbf{u} + \mathbf{a} \mathbf{t}_{A}$$

$$6.67 = 0+0.15 t_A$$

$$t_A = 44.67 \text{ sec.}$$





$$S_1 = u t_A + \frac{1}{2} a t_A^2$$

 $S_1 = 0 + \frac{1}{2} 0.15 \times (44.67)^2$
 $S_1 = 150m$

Since the train B leaves 40 seconds later, so that the train A has travelled (T+40) sec.

.. Distance travelled by train A in (T+60) sec, $S_A = S_1 + V[(T+60) - t_A]$

$$S_A = S_1 + V[(1+60)-t_A]$$

 $S_A = 150+6.67[(T+60)-44.67]$ (1)

Consider the motion of Train B

Initial velocity, u = 0

Final velocity, V = 48 km/hr 48×1000

 $= \frac{48 \times 1000}{3600} = 13.34 \text{ m/s}$ Acceleration, $a = 0.30 \text{ m/s}^2$

 t_B = time taken by train B to attain a speed of 13.34 m/s.

$$V = u + a t_B$$

 $13.34 = 0.33t_B$
 $t_B = 44.47 \text{ sec.}$

Distance travelled by train B in 44.47 sec.

$$S_2 = ut_B + a t_B^2$$

 $S_2 = 0 + \frac{1}{2} 0.5 \times (44.47)^2$
 $S_2 = 296.63m$

.. Distance travelled by train B in T seconds is

$$S_B = S_2 + V(T - t_B)$$





$$S_B = 296.63+13.34(T-44.47)-44.67$$
](2)

At the instent, when train B overtake trains will be equal. Hence

$$S_A = S_B$$

150 + 6.67 [(T+60)-44.67] = 296.63+13.34 (T-44.47)

150 + 6.67T+400.2-297.94 = 296.63+13.34T-593.22

6.67T+252.26 = 13.34T-296.59

6.67T = 548.85

T = 82.28 seconds (Ans)





Car A accelerates uniformly from rest on a straight level road. Car B starting from the same point 6 seconds later with zero initial velocity accelerates at 6m/s². It overtakes the car A at 400m from the starting point. What is the acceleration of the car A? (AU, Apr'11)

Given:

Initial velocity of car A, $u_A = 0$

Initial velocity of car B, $u_B = 0$

acceleration of car B, $a_{\rm B} = 6 \text{m/s}^2$

Distance travelled by car A and car B, $S_A = S_B = 400 m$

To Find:

Acceleration to car A, $a_a = ?$

Solution:

Let 'tA' be the time taken by car 'A'.

Since the car 'B' starts 6 seconds later, the time taken by car B is, $t_{\rm B} = t_{\rm A} - 6$

Consider motion of car 'A'

$$S_A = u_A t_A + \frac{1}{2} a_A t_A^2$$





$$a_A t_A^2 = 800$$

Consider motion of car 'B'

$$S_{B} = u_{B}t_{B} + \frac{1}{2}a_{B}t_{B}^{2}$$

$$400 = 0 + \frac{1}{2} \times 6(t_{A} - 6)^{2}$$

$$\frac{800}{6} = t_{A}^{2} + 36 - 12 t_{A}$$

$$t_A^2 - 12t_A - 97.33 = 0$$

Solving we get, $t_A=17.54$ sec.

Substituting $t_A=17.54$ sec. in eqn. (1)

$$a_A(17.54)^2 = 800$$

$$a_A = 2.6 \text{ m/s}^2$$

(Ans)

..... (1)





A stone is dropped into a well . The sound of the splash is heard 3.63 seconds later. How far below the ground is the surface of water in the well? . Assume the velocity of sound as 331m/s. (AU, Apr'11, Dec '12)

Given:

Velocity of sound, v = 350 m/s. Initial velocity, u = 0

Solution:

Let t = time taken by stone to reach bottom of well Depth of well is

$$h = ut + \frac{1}{2}gt^2$$

$$= 0 + \frac{1}{2} \times 9.81 \times t^2$$

 $h = 4.9 t^2$

We know,

Time taken by sound to reach the top

$$= \frac{\text{Depth of well}}{\text{Velocity of sound}}$$

$$= \frac{h}{350}$$

$$= \frac{4.9 \text{ t}^2}{}$$





It is given that,

Total time taken = 3 seconds

Total time = time taken by stone to reach bottom of well + time taken by sound to reach the top of well.

$$3 = t + \frac{4.9 t^2}{350}$$

$$1050 = 350t + 4.9 t^2$$

$$4.9 t^2 + 350t - 1050 = 0$$

$$t = \frac{-350 \pm \sqrt{(350)^2 - 4 \times 4.9(-1050)}}{2 \times 4.9}$$

$$= \frac{-350 \pm 378.26}{9.8}$$

$$t = 2.9$$
 seconds

Substituting the value of 't' in equation (1) we get,

$$h = 4.9 (2.9)^2$$

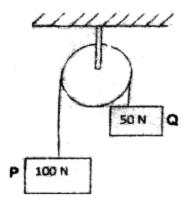
$$h = 41.21m$$

(Ans)





Block P of weight 100N and block Q of weight 50N are connected by a rope that passes over a smooth pulley as shown in figure. Find the acceleration of the blocks and the tension in the rope, when the system is released from rest. Neglect the mass of the pulley. (AU, Apr'11, Dec'12)



Given

Weight of block P, W_P=100N Weight of block, Q, W_O=50N





To find

- 1. Acceleration of blocks, a = ?
- 2. Tension in the rope, T = ?

Solution

Let 'T' be the tension in the string.

Since the weight of block 'P' is larger, it will moves downward and the block 'Q' moves upward.

Considering the motion of block 'Q':-

The various forces acting on block 'Q' is shown in figure.

Resolving forces vertically,

$$\Sigma F_{y} = 0$$

$$T-50-m_{Q}a = 0$$

$$T-50-\frac{50}{9.81} \times a = 0$$

$$T = 50+5.09a$$

$$T = 0$$

$$T$$

Considering the motion of block 'P'

The various forces acting on block 'P' is shown in figure.

Resolving forces vertically,





$$\Sigma F_{y} = 0$$

$$T-50-m_{Q}a = 0$$

$$T-50-\frac{50}{9.81} \times a = 0$$

$$T = 0$$

$$T$$

Considering the motion of block 'P'

T = 50+5.09a

The various forces acting on block 'P' is shown in figure.

Resolving forces vertically,

$$\Sigma F_{y} = 0$$

$$T + m_{p} a - 100 = 0$$

$$T + \frac{100}{0.81} \times a - 100 = 0$$

$$Motion T$$

$$P$$

$$100V$$

$$T = 100 - 10.19a$$
(2

Solving equations (1) & (2) we get

$$50+5.09a = 100-10.19a$$

 $15.28a = 50$
 $a = 3.27 \text{ m/s}^2$ (Ans)

Put a = 3.27 in equation (2), we get

$$T = 100-10.19 (3.27)$$

 $T = 66.67 N$ (Ans)





A 2000 kg automobiles is driven down a 5° inclined plane at a speed of 100km/h when the brakes are applied causing a constant total braking force (applied by the road on the tires) of 7KN.Determine the distance travelled by automobile as it comes to a stop.

(AU, Apr'11, Dec'12)

Given:

Mass of the automobile, m =200kg

$$W = 2000 \times 9.81 = 19620 N = 19.62 kN$$

Initial velocity of car, u = 100km/hr

$$=\frac{100\times1000}{3600}$$
 = 27.78 m/s

Final velocity of car, v = 0

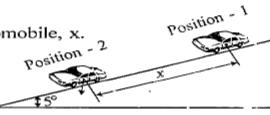
Total breaking force, F = 7 kN

To find:

Distance travelled by automobile, x.

Solution:

To find work done:





Motion

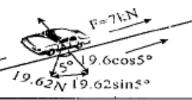


Total work done

$$= [19.62 \sin 5^{\circ} \times x] - [7 \times x]$$

$$= 1.709x - 7x$$

$$= -5.291x$$



To find change in kinetic energy:

Initial Kinetic energy of automobile at position 1,

$$=\frac{1}{2}$$
 mu²

$$=\frac{1}{2}\times2000\times27.78^2$$

$$= 77.1.72 \times 10^3$$
 Joules.

Final Kinetic energy of auotmobile, at position 2,

$$=\frac{1}{2} \text{ mu}^2 = \frac{1}{2} \times 2000 \times 0$$





Change in Kinetic Energy = Final Kinetic Energy-Initial Kinetic Energy

$$= 0-771.72 \times 10^3$$

$$= -771.72 \text{ KJ}.$$

by work energy principle

Total workdone = change in Kinetic energy

$$-5.29/x = -771.72$$

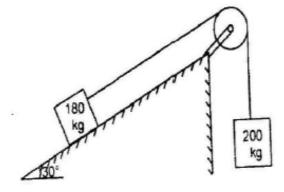
$$x = 145.8 \text{ m}$$

(Ans)





A block and pulley system is shown in fig below. The coefficient of kinetic friction between the block and the plane is 0.25. The pulley is frictionless. Find the acceleration of the blocks and the tension in the string when the system is just released. Also find the time required for 200kg block to come down by 2m. (AU, Jun'09, DEC 11)



Given:

Weight,
$$m_1 = 180 \text{ kg}$$

 $w_1 = 180 \times 9.81 = 1765.8 \text{ N}$
 $m_2 = 200 \text{ kg}$
 $w_2 = 180 \times 9.81 = 1765.8 \text{ N}$

Co-efficient of friction, K = 0.25distance, s = 2m





To Find:

- 1. Acceleration of blocks, a = ?
- 2. Tension is string, T = ?
- 3. Time required, t=?

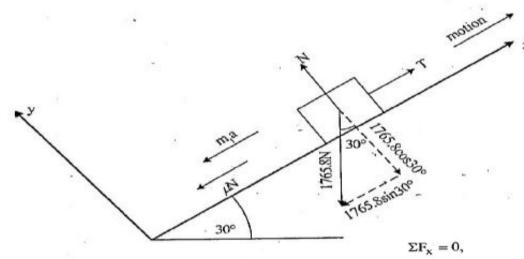
Solution:

1) To find acceleration of blocks:

.Let 'T' be the tension in the string.

Considering the motion of 180 kg block

The forces acting on 180 kg block is shown in the figure.



Resolving forces horizontally,

$$T - m_1 a - \mu N - 1765.8 \sin 30^\circ = 0$$

$$T - 180a - 0.24 N - 882.9 = 0$$





```
Resolving forces vertically,
```

 $\Sigma F_y = 0,$ $N - 1765.8 \cos 30^{\circ};$

 $N = 1703.8 \cos 30^{\circ}$ N = 1529.22 N

Put N = 1529.22 in eqn. (1)

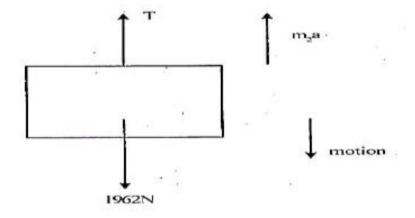
T-180a-0.25(1529.22)-882.9=0

T - 180a = 1265.205

....(2)

....(3)

Considering motion of 200 kg block, The forces acting on 200 kg block is shown in figure.



Resolving forces vertically

 $T + m_2 a - 1962 = 0$ T + 200 a = 1962

Solving eqns. (2) & (3) we get

 $a = 1.833 \text{ m/s}^2$





2) To find tension in the string

Put a = 1.833 in eqn. (3),

T+200 (1.833) = 1962

T = 1595.4 N (Ans)

To find time required.

Since the blocks starts from rest, its initial velocity, u=0 using relation

$$s = ut + \frac{1}{2}at^2$$

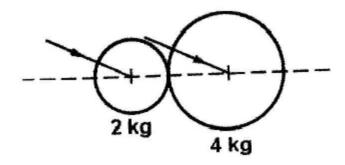
$$2 = 0 + \frac{1}{2} \times 9.81 \times t^2$$

$$t = 0.638$$
 seconds (Ans)





A ball of mass 2kg, moving with a velocity of 3m/s,impinges on a ball of mass 4 kg moving with a velocity of 1m/s. The velocities of the two balls are parallel and inclined at 30° to the line of joining their centres at the instant of impact. If the coefficient of restitution is 0.5, find (AU, Dec'09, Jun'10)



- (i) Direction, in which the 4kg ball will move after impact;
- (ii) Velocity of the 4 kg ball after impact;
- (iii)Direction, in which the 2kg ball will move after impact;
- (iv) Velocity of the 4kg ball after impact.





Given:

Mass of first ball, $m_1 = 2Kg$

Mass of second ball, $m_2 = 4Kg$

Initial velocity of first ball, $U_1 = 3m/s$

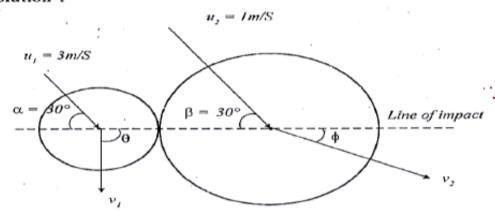
Initial velocity of second ball, $U_2 = m/s$

Angle made by first ball with line of impact, $\alpha = 30^{\circ}$

Angle made by 2^{nd} ball with line of impact, $\beta = 30^{\circ}$

Coefficient of restitution, e = 0.5

Solution:







Let.

Final velocity of first ball after impact = V1

Final velocity of second ball after impact = V2

Angle made by first ball after impact = 6

Angle made by second ball after impact = \$\phi\$

The components of velocity of each ball perpendicular to line of impact before and after impact is same.

For Ball 1,

Vertical component of initial velocity = vertical component of final velocity.

. U₁Sina

 $= V_1 Sin\theta$

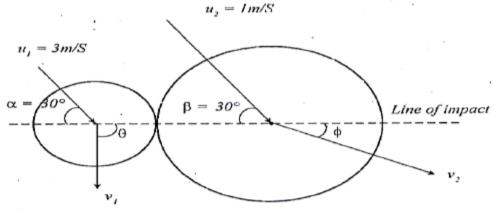
3 Sin30°

 $= V_1 Sin\theta$

Angle made by 2^{nd} ball with line of impact, $\beta = 30^{\circ}$

Coefficient of restitution, e = 0.5

Solution:



Let,

Final velocity of first ball after impact = V1

Final velocity of second ball after impact = V_2

Angle made by first ball after impact = 0

Angle made by second ball after impact = ϕ

The components of velocity of each ball perpendicular to line of impact before and after impact is same.

For Ball 1,

Vertical component of initial velocity = vertical component of final velocity.



V₂Sinф



```
U2SinB
              1 Sin30°
                                      V2Sino
                                                                 ....(2)
              V2Sind
                                      0.5
      By law of conservation of momentum,
Momentum before impact = moment after impact.
       m_1 u_1 \cos \alpha + m_2 u_2 \cos \beta = m_1 v_1 \cos \theta + m_2 v_2 \cos \phi
       2\times3\times\cos30^{\circ} + 4\times1\times\cos30^{\circ} = 2v_{1}\cos\theta + 4v_{2}\cos\phi.
       2v_1\cos\theta + 4v_2\cos\phi = 8.66
                                                                  ....(3)
       v_1 \cos\theta + 2v_2 \cos\phi = 4.33
       Co-efficient of restitution is
                                         v_2 \cos \phi - v_1 \cos \theta
                                        u_1 \cos \alpha - u_2 \cos \beta
```

 $0.5 [3 \times 0.866 - 1 \times 0.866] = v_2 \cos \phi - v_1 \cos \theta$...(4) = 0.866 $v_2 \cos \phi - v_1 \cos \theta$

Adding equations (3) & (4), we get

0.5

 $v_2 \cos \phi - v_1 \cos \theta$

 $3\cos 30^{\circ} - 1\cos 30^{\circ}$





$$3v_2 \cos \phi = 5.196$$

 $v_2 \cos \phi = 1.732$ (5)

= 16.1

i) To find direction in which 4kg ball move after impact:

Dividing equations (2) by (5)

$$\frac{(2)}{(5)} \Rightarrow \frac{\mathbf{v}_2 \sin \phi}{\mathbf{v}_2 \cos \phi} = \frac{0.5}{1.732}$$

ii) To find velocity of 4Kg ball after impact,

Put
$$\phi = 16.1$$
 in eqn. (2),

$$v_2 \sin 16.1 = 0.5$$

tano

$${
m v}_2=1.803~{
m m/sec.}$$
 (Ans)
To find direction in which 2 Kg. ball move after impact

(Ans)

(Ans)

impact, substituting the values of ϕ and V_2 in eqn. (4)

$$1.803\cos 16.1 - v_1\cos\theta = 0.866$$

$$v_1 \cos \theta = 1.803 \cos 16.1 - 0.866$$

$$v_1 \cos \theta = 0.866 \qquad \dots (6)$$

$$\frac{(1)}{(6)} \Rightarrow \frac{\mathbf{v}_1 \sin \theta}{\mathbf{v}_1 \cos \theta} = \frac{1.5}{0.866}$$

$$\tan \theta = 1.732$$





iv) To find velocity of 2kg ball after impact.

Substitute
$$\theta = 60^{\circ}$$
 in eqn. (1),

$$v_1 \sin 60^{\circ} = 1.5$$

$$v_1 = 1.732 \text{ m/sec.}$$
 (Ans)



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QPQA 14 marks questions



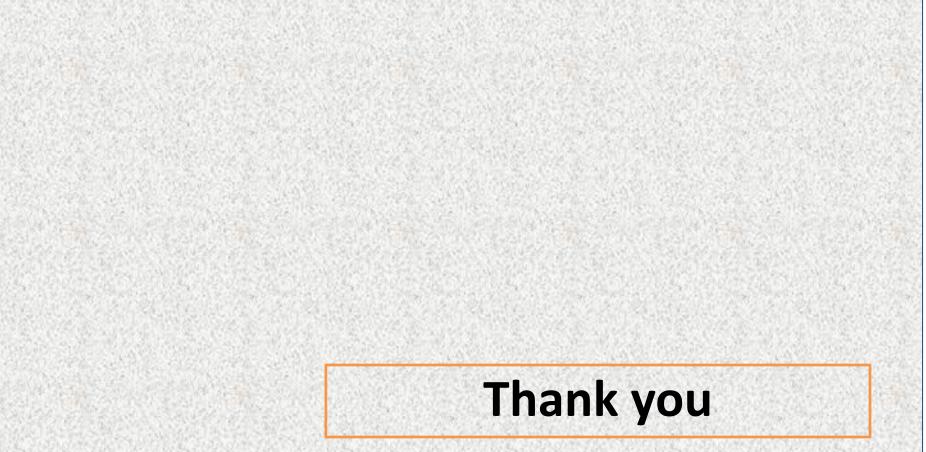
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16ME 102 /Engineering Mechanics







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16ME 102 /Engineering Mechanics

90/90