

1.) Max. principle (or) normal stress theory (or) Max. stress theory

(or) Rankine's theory:-

According to this theory, The failure (or) yielding ~~stress~~ occurs at a point in a member when the maximum principle (or) normal stress in a bi-axial stress system reaches the limiting strength of the material in a simple tension test.

$$\sigma_1 \text{ (or) } \sigma_2 \text{ (or) } \sigma_3 \text{ (whichever is max)} = \sigma_y$$

$$\text{Max. Normal Stress} = \frac{\sigma_{yt}}{F.S} \text{ (Ductile)}$$

$$\sigma_t = \frac{\sigma_u}{F.S} \text{ [Brittle]}$$

for design

$$\sigma_1 \text{ (or) } \sigma_2 \text{ (or) } \sigma_3 = \frac{\sigma_y}{n}$$

where, $n = \text{Factor of safety}$

$$\sigma_1 \text{ (or) } \sigma_2 \text{ (or) } \sigma_3 = \frac{\sigma_{yt}}{n} \text{ [For Brittle material]}$$

This theory is best suited for brittle material.

2.) Max. Shear Stress theory (or) Max. shear theory (or)

Guest's theory:-

Acc. to this theory, the failure (or) yielding occurs at a point in a member when the max. shear stress in a bi-axial stress system reaches a value equal to the shear stress at yield point in a simple tension test.

$$\tau_{\text{max}} = \frac{\tau_{yt}}{F.S}$$

← Shear for yield point

$$(\sigma_1 - \sigma_2) \text{ (or) } (\sigma_2 - \sigma_3) \text{ (or) } (\sigma_3 - \sigma_1) \text{ } \left. \vphantom{(\sigma_1 - \sigma_2)} \right\} = \frac{\sigma_y}{n}$$

whichever is max.

Problems:

1) A bolt is subjected to a tensile load of 25kN and shear load of 10kN. determine the diameter of bolt acceleration to

- i) Maximum principle stress theory.
- ii) Maximum principle strain theory.
- iii) Maximum shear stress theory.

• Assume the factor of safety as 2.5.
 Yield point stress = 300 N/mm².
 Poisson's ratio = 0.25.

Given data:-

$P = 25 \text{ kN} = 25 \times 10^3 \text{ N}$

Shear load (P_s) = 10kN = $10 \times 10^3 = 10 \times 10^3 \text{ N}$

Yield point stress (σ_y) = 300 N/mm²

Poisson's ratio ($\frac{1}{m}$) = ($\frac{\nu}{2}$) = 0.25
FOS = 2.5

To find:-

$\sigma_x = ?$
 $\tau = ?$

Solution:- Axial tensile stress

$\sigma_x = \sigma_t = \frac{P}{A} = \frac{25 \times 10^3}{\frac{\pi}{4} d^2} = \frac{31.83 \times 10^3}{d^2}$

transverse shear stress

$\tau = \frac{P_s}{A} = \frac{10 \times 10^3}{\frac{\pi}{4} d^2} = \frac{12.73 \times 10^3}{d^2}$

$\sigma_x = \sigma_t$

$\tau_{xy} = \tau$ $\sigma_y = 0$

From D.D.B Pg-no. 7.2

$$\sigma_{1,2} = \frac{1}{2} [\sigma_x + \sigma_y] \pm \sqrt{(\sigma_x - \sigma_y)^2 + 4\tau_{xy}^2}$$

Maximum principle stress:-

$$\sigma_1 = \frac{1}{2} [\sigma_x + \sigma_y] + \sqrt{(\sigma_x - \sigma_y)^2 + 4\tau_{xy}^2}$$

$$\sigma_1 = \frac{1}{2} \left[\frac{31.83 \times 10^3}{d^2} + \sqrt{\left(\frac{31.83 \times 10^3}{d^2}\right)^2 + 4 \left[\frac{12.73 \times 10^3}{d^2}\right]^2} \right]$$

$$\sigma_1 = \frac{1}{2} \left[\frac{31.83 \times 10^3}{d^2} + \frac{1}{d^2} \sqrt{(31.83 \times 10^3)^2 + 4(12.73 \times 10^3)^2} \right]$$

$$\sigma_1 = \frac{1}{2} \left[\frac{31.83 \times 10^3}{d^2} + \frac{40.76 \times 10^3}{d^2} \right]$$

$$\sigma_1 = \frac{1}{2} \left[\frac{72.59 \times 10^3}{d^2} \right]$$

$$\sigma_1 = \frac{36.295 \times 10^3}{d^2}$$

Minimum principle stress

$$\sigma_2 = \frac{1}{2} [\sigma_x + \sigma_y] - \sqrt{(\sigma_x - \sigma_y)^2 + 4\tau_{xy}^2}$$

$$\sigma_2 = \frac{1}{2} \left[\frac{31.83 \times 10^3}{d^2} - \frac{40.76 \times 10^3}{d^2} \right]$$

$$\sigma_2 = -\frac{4.446 \times 10^3}{d^2}$$

$$d = 17.39 \text{ mm}$$

$\therefore \sigma_1 = \text{larger value}$

$$\sigma_1 = \frac{36.295 \times 10^3}{d^2}$$

$$= \frac{36.295 \times 10^3}{d^2} = \frac{300}{2.5}$$

FoS = $\frac{\sigma_1}{\sigma_{\text{Yield stress (or) max. stress}}}$

σ_1 = Allowable stress (or) Permissible stress

$\sigma_1 = \frac{\sigma_{\text{Yield stress (or) max. stress}}}{\text{FoS}}$

Maximum principal strain theory:-

$$\sigma_1 = \delta [\sigma_2 + \sigma_3] = \frac{\sigma_y \leftarrow \text{Yield stress}}{n \leftarrow \text{FOS}} \quad (\text{From P.S.G DPB Pg. no 7.3})$$

$$\frac{36.295 \times 10^3}{d^2} - 0.25 \left[\frac{-4.446 \times 10^3}{d^2} \right] = \frac{300}{2.5}$$

$$\frac{36.295 \times 10^3}{d^2} + \frac{1.11 \times 10^3}{d^2} = 120$$

$$\boxed{d = 17.65 \text{ mm}}$$

Maximum shear stress theory:-

(DDB Pg. no 7.3)

$$\text{Maximum shear stress} = \sigma_1 - \sigma_2 = \frac{\sigma_y}{n} \text{ (or)} \frac{\sigma_y}{\text{FOS}}$$

$$= \frac{36.295 \times 10^3}{d^2} - \left[\frac{-4.446 \times 10^3}{d^2} \right] = \frac{300}{2.5}$$

$$= \frac{40.71 \times 10^3}{d^2} = \frac{300}{2.5}$$

$$\frac{40.71 \times 10^3}{d^2} = \frac{300}{2.5} \Rightarrow \boxed{d = 18.43 \text{ mm}}$$

$\boxed{\text{Major dia of bolt is } 18.43 \text{ mm}}$